

22.314/1.56/2.084/13.14 Fall 2006
Problem Set I Solution

1. Original dimension:

$$D_0 = 12.8 \text{ mm}, A_0 = \pi D_0^2 / 4 = 128.68 \text{ mm}^2; \text{ and } L_0 = 50.800 \text{ mm.}$$

- (a) Stress at a load $F = 22.2 \text{ kN}$:

$$\sigma = F/A_0 = 172.5 \text{ MPa}$$

Strain:

$$\epsilon = (L - L_0)/L_0 = (50.848 - 50.8)/50.8 = 0.009448$$

Young's Modulus:

$$E = \sigma/\epsilon = 182.6 \text{ GPa}$$

- (b) Maximum normal strain is the strain when fracture occurs:

$$(L_{max} - L_0)/L_0 = (69.8 - 50.8)/50.8 = 0.374$$

- (c) $F_{max} = 51.2 \text{ MPa}$. Tensile strength is:

$$F_{max}/A_0 = 397.9 \text{ MPa}$$

2. This problem has a stress tensor:

$$\sigma = \begin{bmatrix} \sigma_{xx} & 0 & 0 \\ 0 & \sigma_{yy} & \tau_{yz} \\ 0 & \tau_{zy} & \sigma_{zz} \end{bmatrix}$$

Note that the shear stresses on the plane normal to x direction are zero. Therefore, we can derive the two principal stresses on the yz plane using the solution for a plane stress condition.

$$\sigma_{i,j} = \frac{\sigma_{yy} + \sigma_{zz}}{2} \pm \sqrt{\left(\frac{\sigma_{yy} - \sigma_{zz}}{2}\right)^2 + \tau_{yz}^2}$$

$$\sigma_i = 464.5 \text{ MPa} \text{ and } \sigma_j = 40.5 \text{ MPa.}$$

Therefore, $\sigma_1 = 464.5 \text{ MPa}$, $\sigma_2 = 440 \text{ MPa}$, and $\sigma_3 = 40.5 \text{ MPa}$.

The maximum normal stress is $\sigma_1 = 464.5 \text{ MPa}$.

The maximum shear stress is $(\sigma_1 - \sigma_3)/2 = 212 \text{ MPa}$.

3. (a) The principal stresses are the eigenvalues of the stress tensor. It's solved by using MATLAB (See the code in the end). For

$$\sigma_a = \begin{bmatrix} 55 & -5 & 30 \\ -5 & 55 & 30 \\ 30 & 30 & 20 \end{bmatrix},$$

the principal stresses are found to be:

$\sigma_1 = 80 \text{ MPa}$, $\sigma_2 = 60 \text{ MPa}$, and $\sigma_3 = -10 \text{ MPa}$.

For

$$\sigma_b = \begin{bmatrix} -10 & 0 & 0 \\ 0 & -10 & 0 \\ 0 & 0 & -10 \end{bmatrix},$$

the principal stresses are found to be:

$$\sigma_1 = \sigma_2 = \sigma_3 = -10 \text{ MPa}.$$

For

$$\sigma_c = \sigma_a + \sigma_b,$$

the principal stresses are found to be:

$$\sigma_1 = 70 \text{ MPa}, \sigma_2 = 50 \text{ MPa}, \text{ and } \sigma_3 = -20 \text{ MPa}.$$

- (b) See Figure 1 and Figure 2. The Mohr's Circle for σ_b is a single point (-10, 0).
- (c) The points on the 3-D Mohr's circles that give the stresses on planes normal to each of the original coordinate axes (x, y, z) are:
 $(\sigma_x, \sqrt{\tau_{xy}^2 + \tau_{xz}^2})$, $(\sigma_y, \sqrt{\tau_{yx}^2 + \tau_{yz}^2})$, and $(\sigma_z, \sqrt{\tau_{zx}^2 + \tau_{zy}^2})$
as shown in Figure 1 and Figure 2.

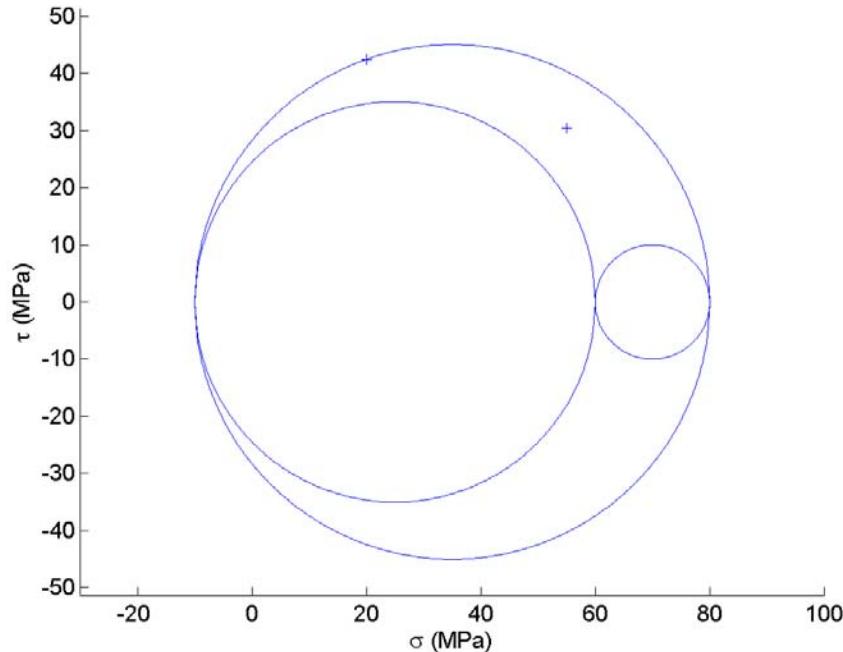


Figure 1: 3-D Mohr's Circle for stress tensor σ_a

- (d) The unit vectors are corresponding eigenvector of each eigenvalue (principal stress) of the stress tensor. For σ_a , the unit vectors are as follows:

$$u_1 = \begin{bmatrix} 0.5774 \\ 0.5774 \\ 0.5474 \end{bmatrix}; u_2 = \begin{bmatrix} 0.7071 \\ -0.7071 \\ 0 \end{bmatrix}; u_3 = \begin{bmatrix} 0.4082 \\ 0.4082 \\ -0.8165 \end{bmatrix}.$$

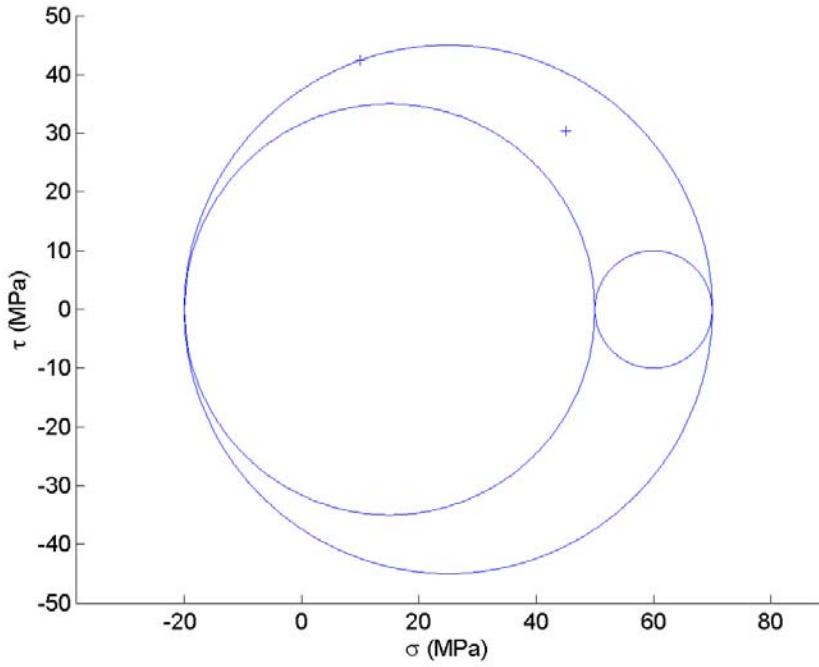


Figure 2: 3-D Mohr's Circle for stress tensor σ_c

For σ_b :

$$u_1 = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}; u_2 = \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix}; u_3 = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}.$$

Since σ_c is the sum of σ_a and hydrostatic pressure tensor σ_b , the unit vectors should be the same as those of σ_a .

$$u_1 = \begin{bmatrix} 0.5774 \\ 0.5774 \\ 0.5474 \end{bmatrix}; u_2 = \begin{bmatrix} 0.7071 \\ -0.7071 \\ 0 \end{bmatrix}; u_3 = \begin{bmatrix} 0.4082 \\ 0.4082 \\ -0.8165 \end{bmatrix}.$$

(e)

$$\sigma_{Tresca} = \max\{|\sigma_1 - \sigma_2|, |\sigma_2 - \sigma_3|, |\sigma_3 - \sigma_1|\}$$

$$\sigma_{vonMises} = \sqrt{\frac{(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2}{2}}$$

Therefore:

	σ_a	σ_b	σ_c
σ_{Tresca}	90 (MPa)	0 (MPa)	90 (MPa)
$\sigma_{vonMises}$	81.85 (MPa)	0 (MPa)	81.85 (MPa)

The MATLAB code to solve Problem 3:

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%=====
%      22.314 (Fall 06) Problem Set I-3.
%      09/19/2006
%
%=====
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Sigma_a = [55 -5 30
 -5 55 30
 30 30 20];
Sigma_b = [-10 0 0
 0 -10 0
 0 0 -10];
Sigma_c = Sigma_a + Sigma_b;

% eig returns eigenvalues and eigenvectors of a matrix
[Eigvec_a Eigval_a] = eig(Sigma_a);
[Eigvec_b Eigval_b] = eig(Sigma_b);
[Eigvec_c Eigval_c] = eig(Sigma_c);

% Mohr's Circle for Sigma_a
s1 = Eigval_a(3,3);
s2 = Eigval_a(2,2);
s3 = Eigval_a(1,1);
c1 = (s1 + s2)/2; r1 = (s1 - s2)/2;
c2 = (s2 + s3)/2; r2 = (s2 - s3)/2;
c3 = (s1 + s3)/2; r3 = (s1 - s3)/2;

x1 = linspace(s2, s1, 200);
y1p = sqrt(r1^2 - (x1 - c1).^2); y1n = -y1p;
x2 = linspace(s3, s2, 200);
y2p = sqrt(r2^2 - (x2 - c2).^2); y2n = -y2p;
x3 = linspace(s3, s1, 500);
y3p = sqrt(r3^2 - (x3 - c3).^2); y3n = -y3p;

figure; hold on;
axis([-30 100 -50 50]);
axis equal;
plot(x1, y1p);
plot(x1, y1n);
plot(x2, y2p);
plot(x2, y2n);
plot(x3, y3p);
plot(x3, y3n);
sigma_x = Sigma_a(1,1);

```

tau_x = sqrt(Sigma_a(1,2)^2 + Sigma_a(1,3)^2);
sigma_y = Sigma_a(2,2);
tau_y = sqrt(Sigma_a(2,1)^2 + Sigma_a(2,3)^2);
sigma_z = Sigma_a(3,3);
tau_z = sqrt(Sigma_a(3,1)^2 + Sigma_a(3,2)^2);

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plot(sigma_x, tau_x, '+');
plot(sigma_y, tau_y, '+');
plot(sigma_z, tau_z, '+');
xlabel('sigma (MPa)')
ylabel('tau (MPa)')

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```

%Mohr's Circle for Sigma_c;
s1 = Eigval_c(3,3);
s2 = Eigval_c(2,2);
s3 = Eigval_c(1,1);

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c1 = (s1 + s2)/2; r1 = (s1 - s2)/2;
c2 = (s2 + s3)/2; r2 = (s2 - s3)/2;
c3 = (s1 + s3)/2; r3 = (s1 - s3)/2;

```

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x1 = linspace(s2, s1, 200);
y1p = sqrt(r1^2 - (x1 - c1).^2); y1n = -y1p;
x2 = linspace(s3, s2, 200);
y2p = sqrt(r2^2 - (x2 - c2).^2); y2n = -y2p;
x3 = linspace(s3, s1, 500);
y3p = sqrt(r3^2 - (x3 - c3).^2); y3n = -y3p;

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figure; hold on;
axis([-20 100 -50 50]);
axis equal;
plot(x1, y1p);
plot(x1, y1n);
plot(x2, y2p);
plot(x2, y2n);
plot(x3, y3p);
plot(x3, y3n);

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sigma_x = Sigma_c(1,1);
tau_x = sqrt(Sigma_c(1,2)^2 + Sigma_c(1,3)^2);
sigma_y = Sigma_c(2,2);
tau_y = sqrt(Sigma_c(2,1)^2 + Sigma_c(2,3)^2);
sigma_z = Sigma_c(3,3);
tau_z = sqrt(Sigma_c(3,1)^2 + Sigma_c(3,2)^2);

```

```
plot(sigma_x, tau_x, '+');  
plot(sigma_y, tau_y, '+');  
plot(sigma_z, tau_z, '+');  
xlabel(' \sigma (MPa) ')  
ylabel(' \tau (MPa) ')
```

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