

Fossil Fuels I

Sustainable Energy

Fall 2010

10/14/2010

Scope of this Session

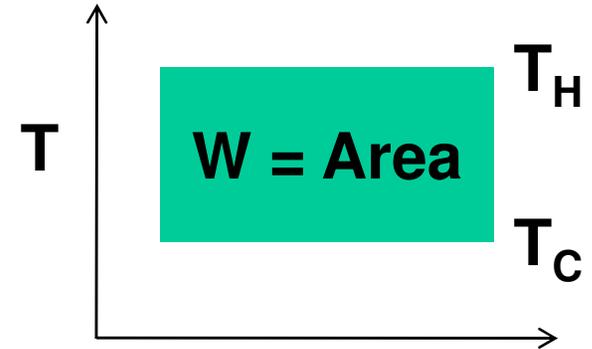
- Cover the major power cycles for conversion of fossil fuels to electricity
 - Steam Cycles
 - Steam Turbines
 - Brayton Cycle
 - Gas Turbines
 - Combined Cycles

Understanding Steam Cycles

- Start with Carnot efficiency as upper limit
- Use reality to chip away at the efficiency
- Use tricks to maximize efficiency

Carnot Efficiency

$$\eta_{\text{Carnot}} \equiv \frac{\dot{W}_{\text{max}}}{\dot{Q}_H} = 1 - \frac{T_C}{T_H}$$



- Assumes Q_H is all available at T_H
- Assumes Q_C is all available at T_C
- Assumes Reversibility
 - No temperature driving force on heat exchangers
 - No pressure drops in exchangers or pipes
 - No entropy losses on turbines or pumps
- For $T_H=1800$ K, $T_C=300$ K, $\eta_{\text{Carnot}}=83\%$

Reality 1

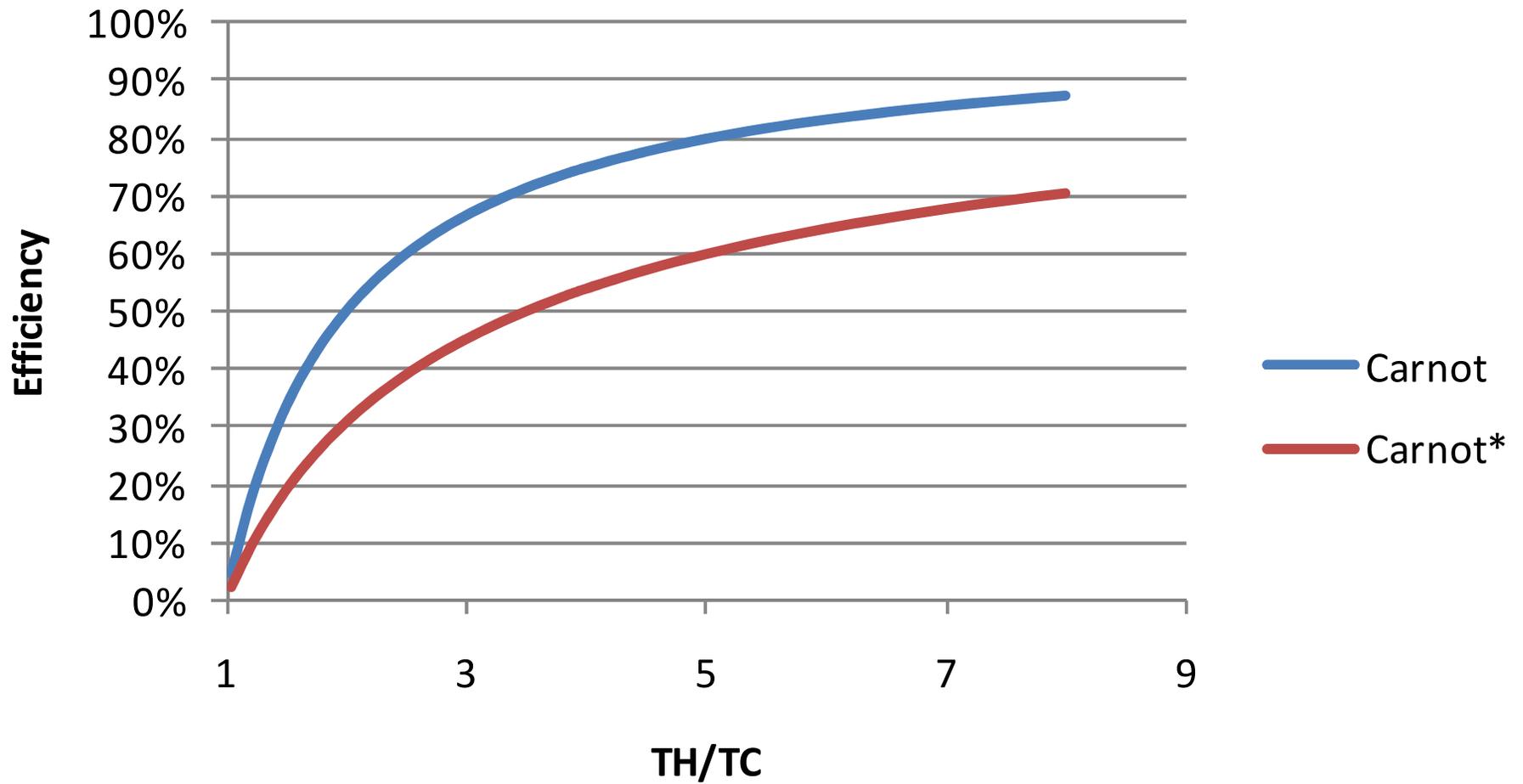
Heat Source Temperature not Constant

- Heat source may start at T_H but the temperature drops as heat is delivered
- Heat is maximized if hot medium exits at T_C
- Maximum work determined by integrating over this temperature profile (assume constant C_p)

$$\eta^*_{\text{Carnot}} = 1 - \ln\left(\frac{T_H}{T_C}\right) / \left(\frac{T_H}{T_C} - 1\right)$$

- For $T_H=1800$ K, $T_C=300$ K, $\eta^*_{\text{Carnot}}=64\%$

Carnot & Carnot* Efficiency for Range of Temperature Ratios

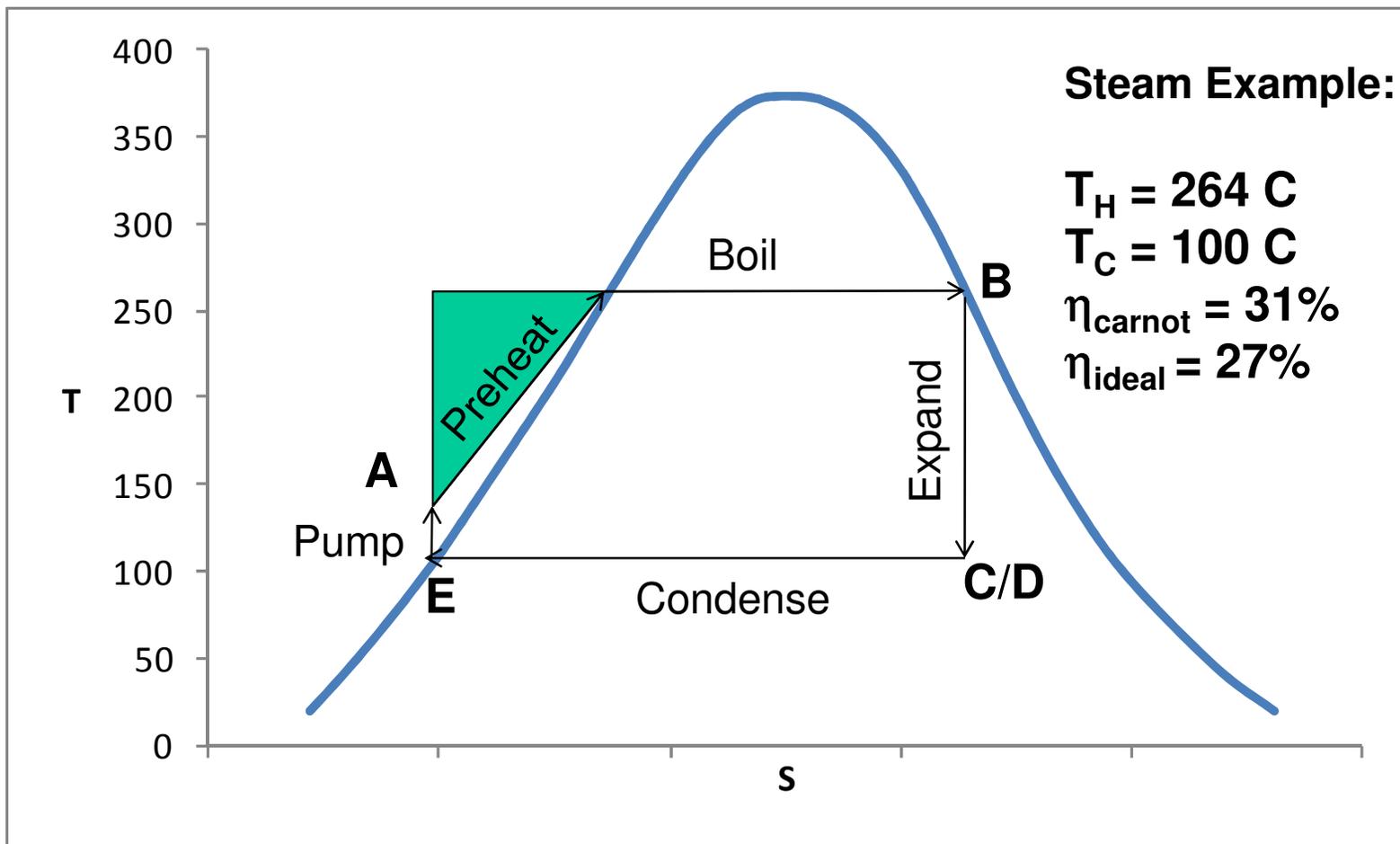


Rankine cycle

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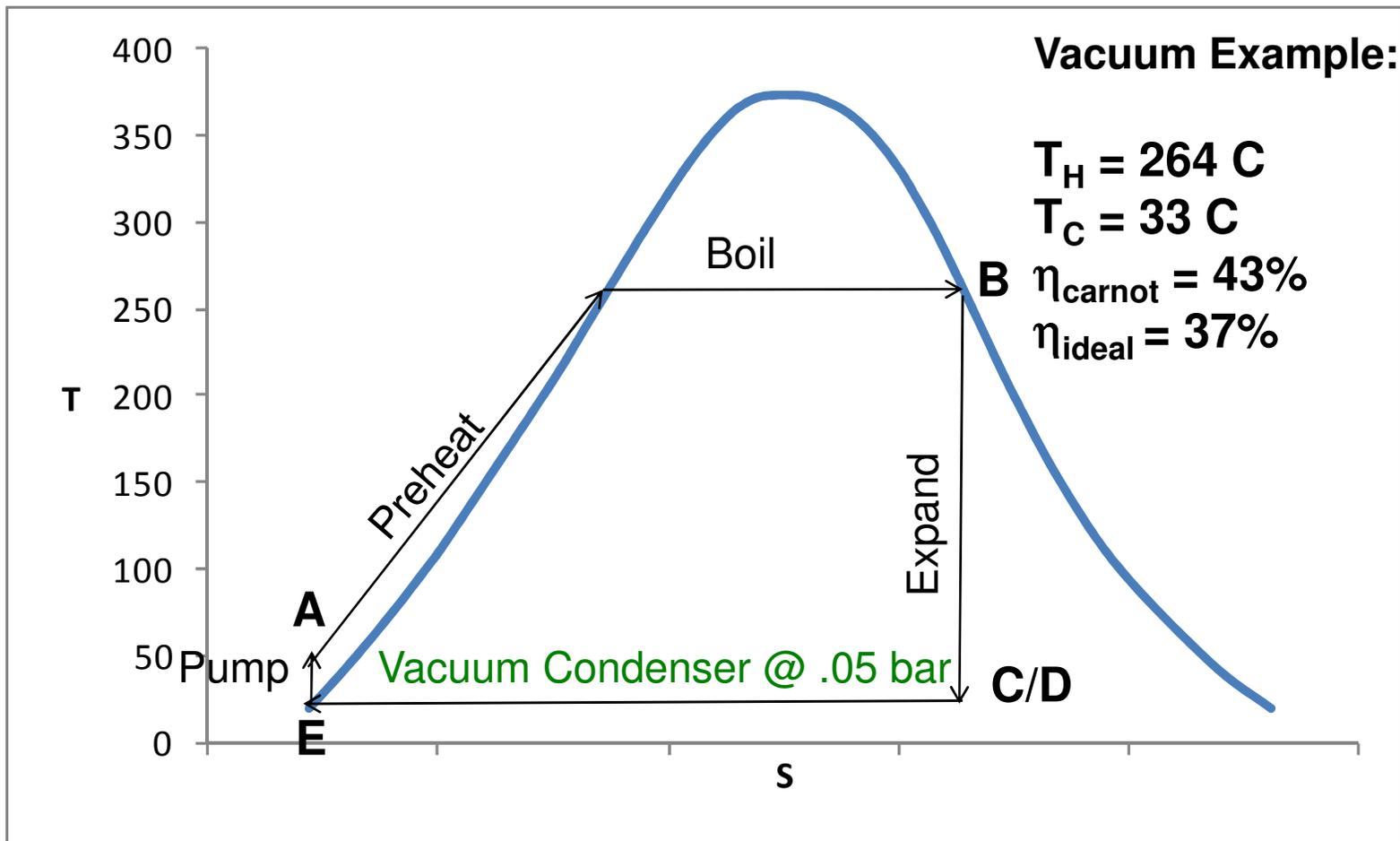
Reality 2

Working Fluid Phase Envelope Matters



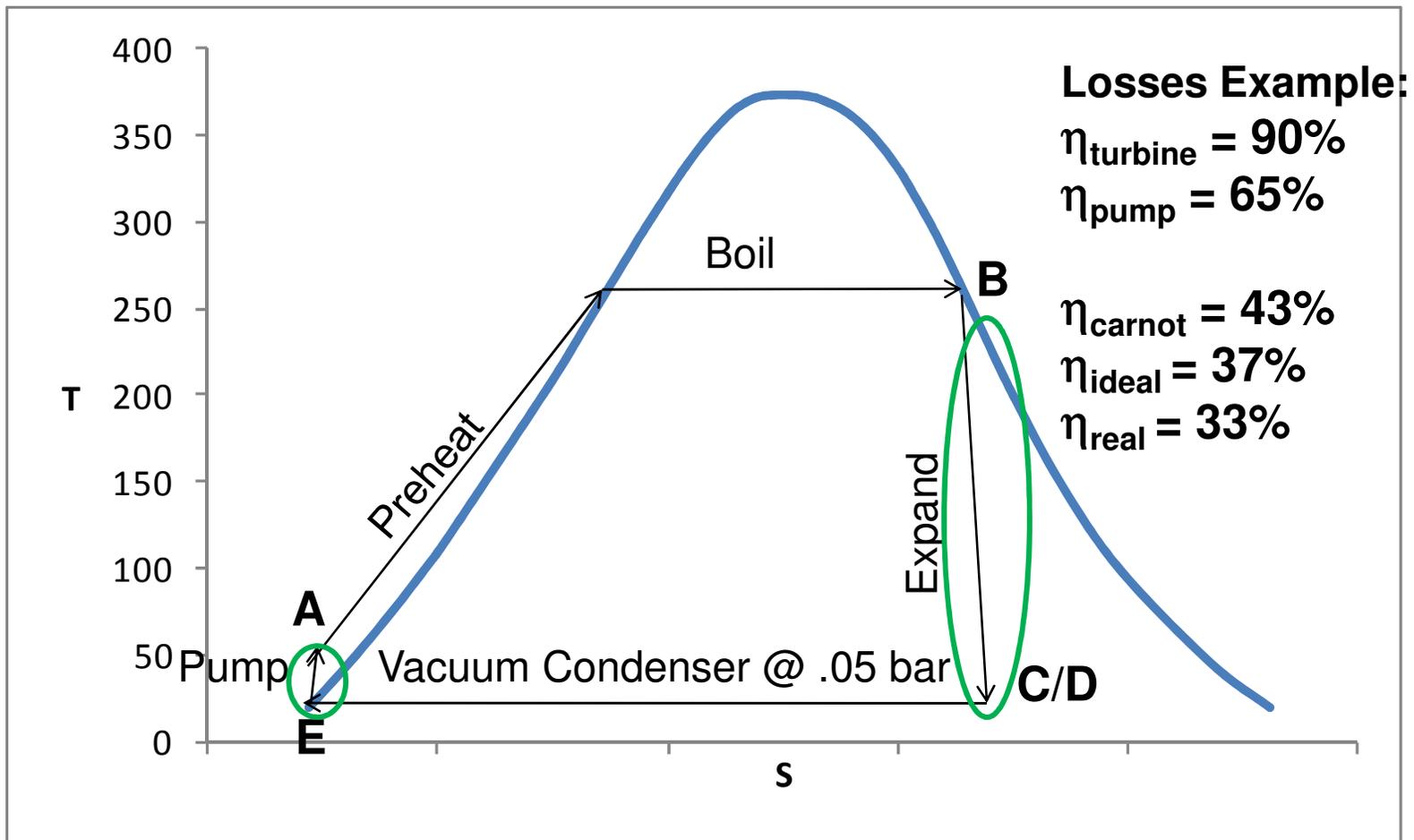
Reality 3

Ambient Pressure not Hard Limit For Closed Cycles



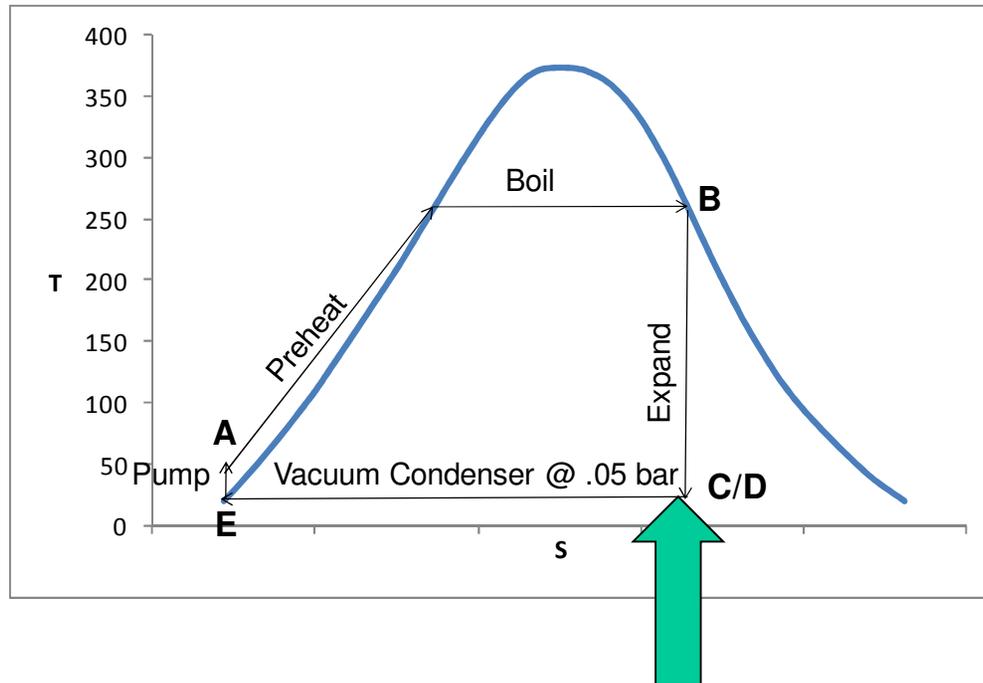
Reality 4

Real Pumps and Turbines have Entropy Losses



Reality 5

Expanding into Two-Phase Region is a Problem



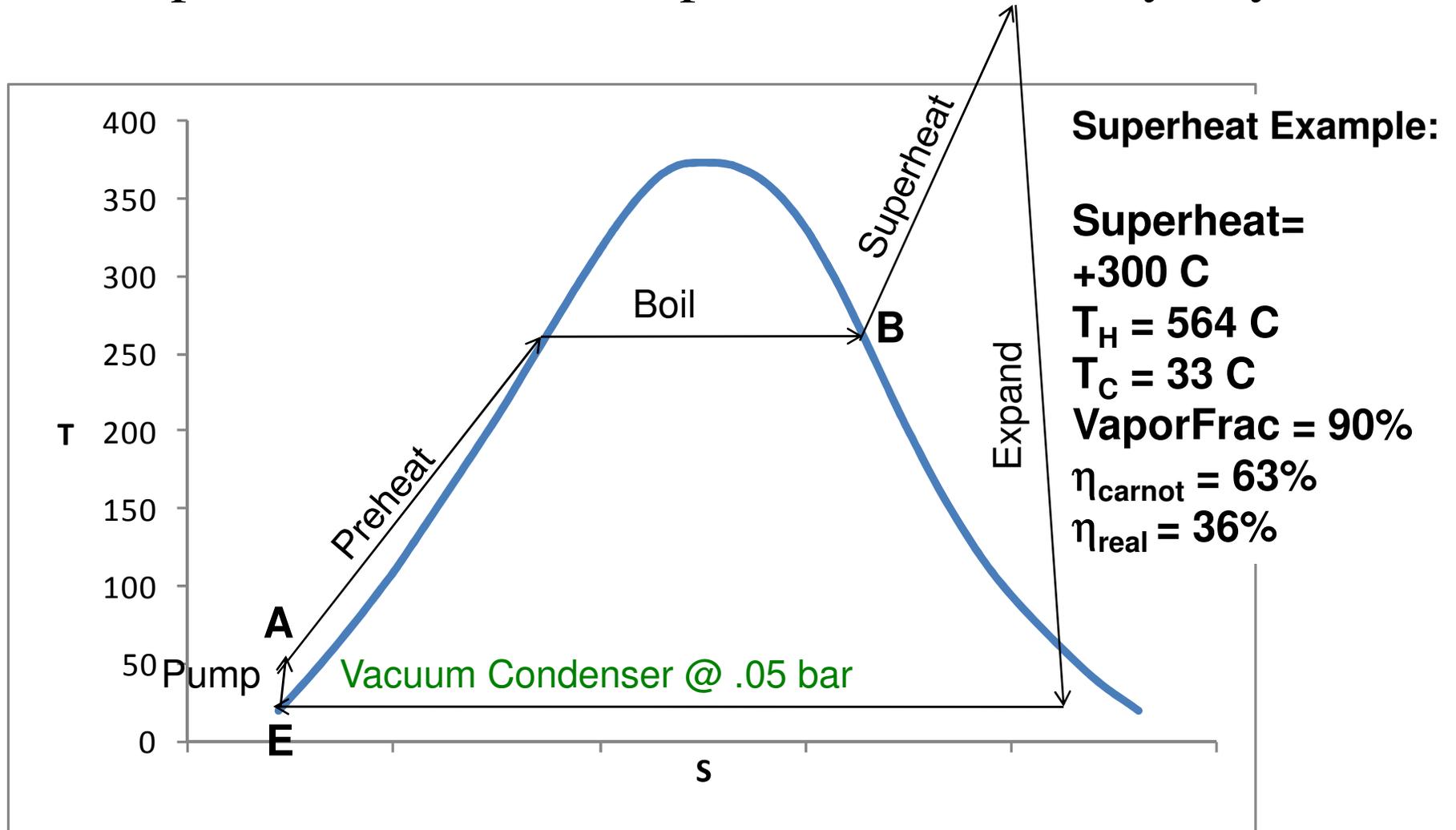
Turbine Exit Vapor Fraction is only 73%

Turbine Reality:

- Vapor fraction must exceed 90%
- Efficiency diminished by condensation in turbine

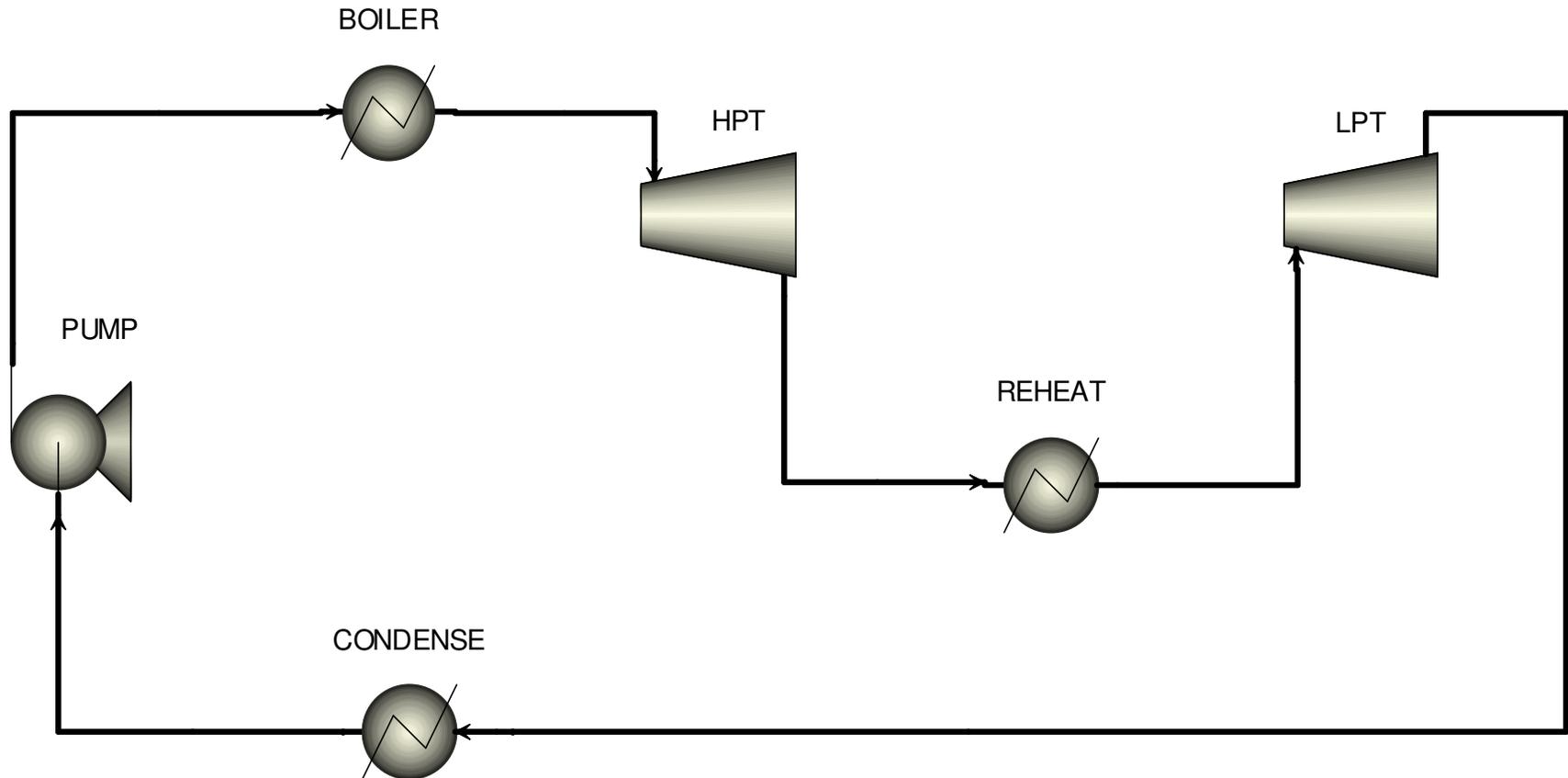
Superheat Cycles

Superheat Steam to Keep Turbine Relatively Dry



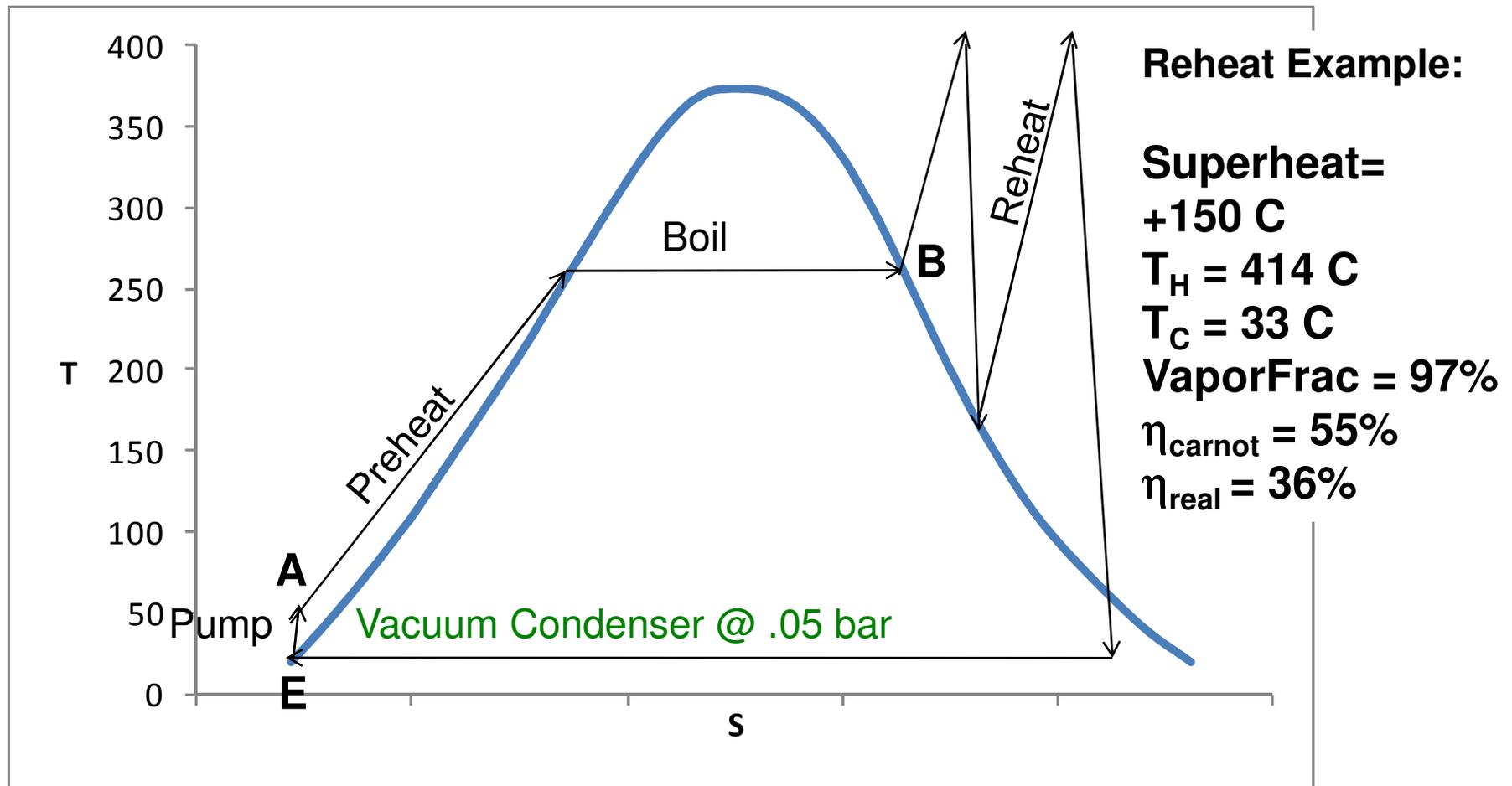
Reheat Cycles

Reheat between Turbines → More Power & Dry Turbines



Reheat Cycles

Reheat between Turbines → More Power & Dry Turbines



Regenerative Cycles

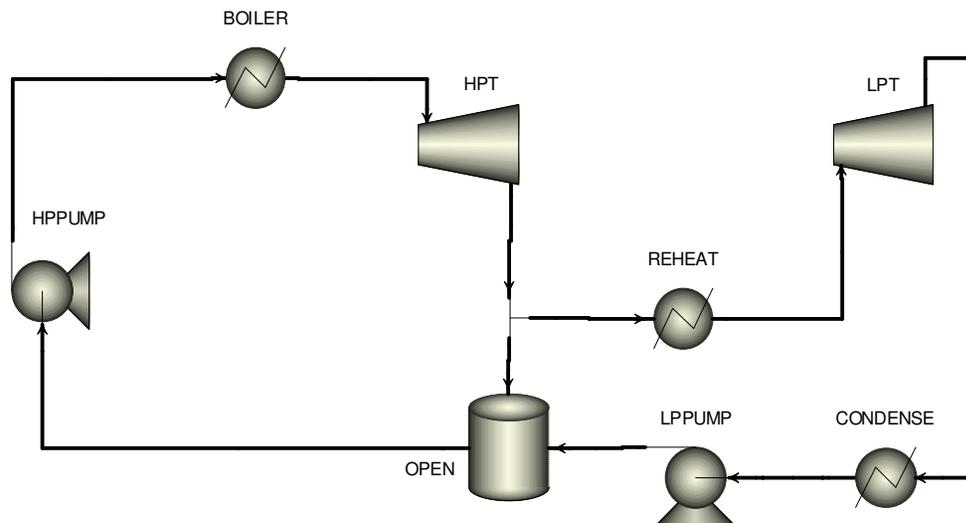
- Preheat with lower quality heat
 - Extract steam from turbines
 - Feedwater heaters
 - Open (Direct contact)
 - Closed (Indirect)

Regeneration Example:

Extraction Factor = 12%

$\eta_{\text{carnot}} = 55\%$

$\eta_{\text{real}} = 37\%$



Real Steam Cycles

- Multiple steam pressure levels
- Multiple reheats
- Multiple extractions for feedwater heating
- Deaerator for oxygen removal
- Best performance at steam pressures $> P_c$
- Maximum steam temperature: $\sim 600^\circ\text{C}$
- Economizer to recover heat from flue gas
- Fuel utilization is key metric for fossil fuel power

$$\eta_{\text{Fuel Utilization}} = \frac{W}{\text{FuelFlow} \times \text{LHV}}$$

Steam Rankine Cycle Summary

- Fuel flexible: works well with coal and other dirty fuels (closed cycle)
- Workhorse for nuclear and most solar thermal
- Low flow rate: thanks to high heat of vaporization
- Low pumping power
- But...
 - Limited by maximum steam temperatures due to material of construction constraints
 - High inertia: good for base load, not for load following
 - Requires cooling: a water hog for many power plants

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Siemens SST-500

Photo of a [Siemens SST-500 steam turbine](#) removed due to copyright restrictions.

Power output: up to 100 MW

Rotational speed: up to 15,000 rpm

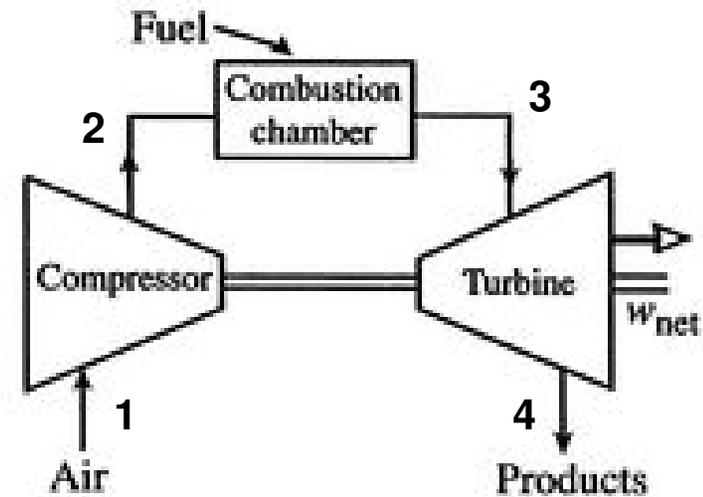
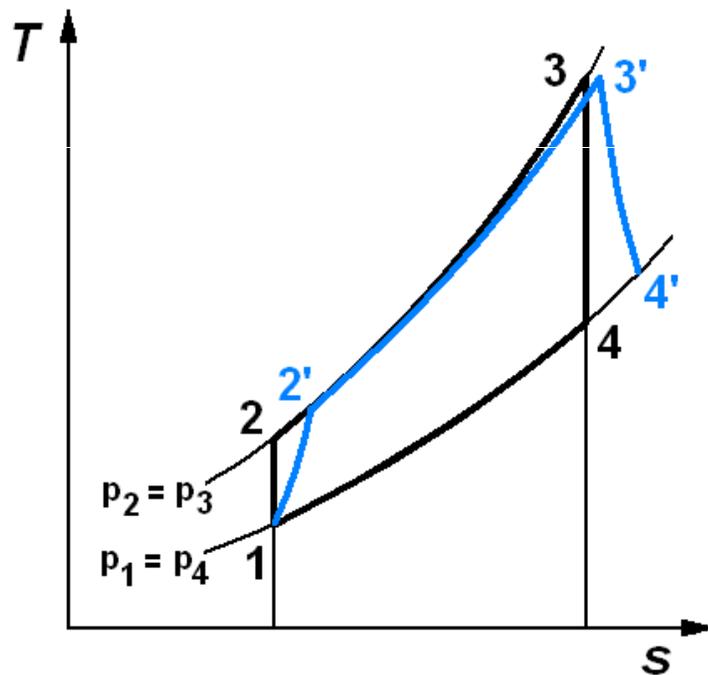
Inlet steam pressure: up to 30 bar

Inlet steam temperature: up to 400 °C

Bleeds: up to 2, at various pressure levels

Brayton Cycle

Brayton Cycle = Rankine Cycle – Boiling – Condensation



Evolution of Turbine Blade Technology

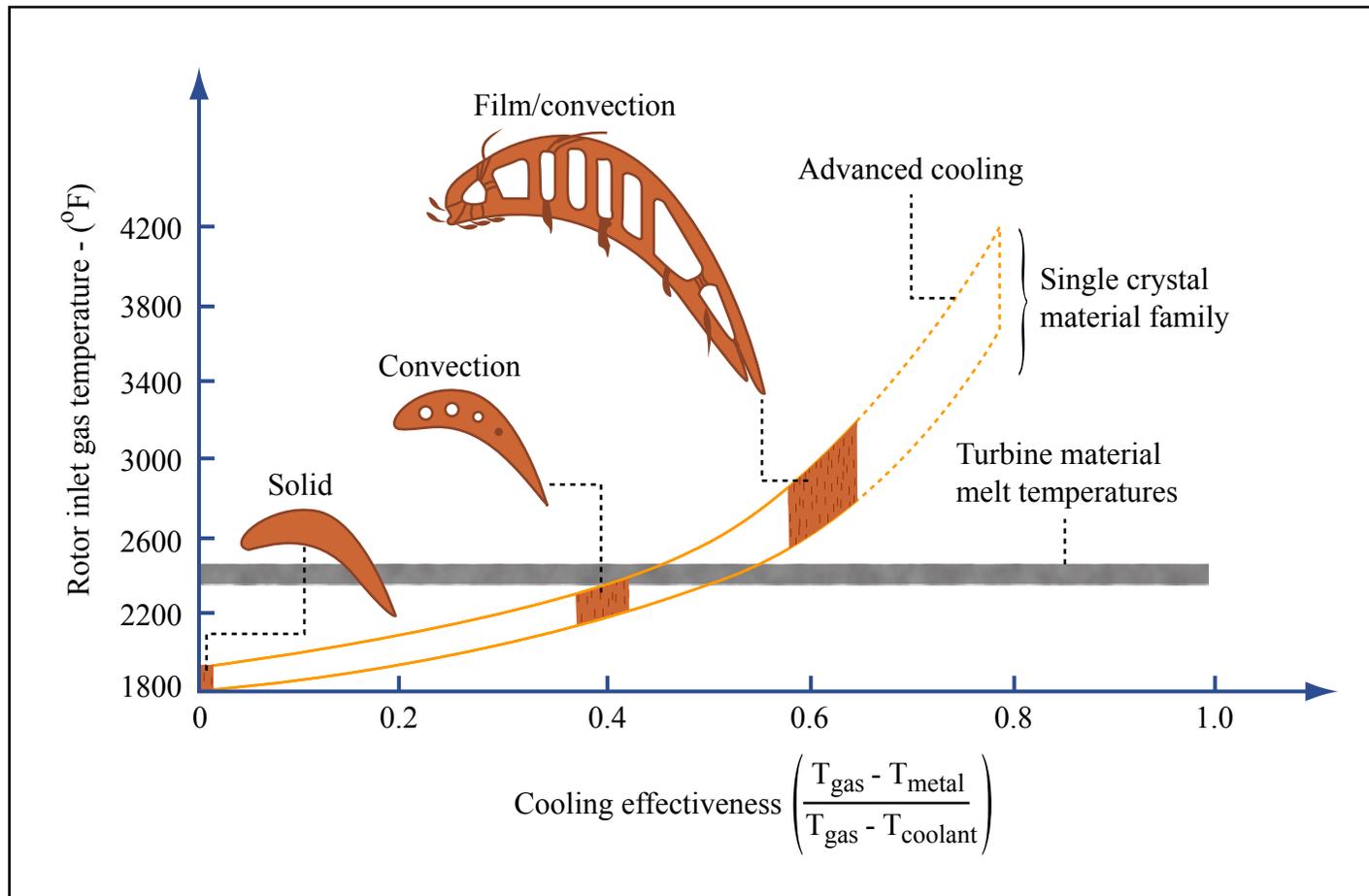


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Source: MIT Unified Engineering 16.003/16.004

Siemens Gas Turbine SGT5-8000H

Photo of a [Siemens SGT5-8000H gas turbine](#) removed due to copyright restrictions.

Power Output: 375 MW
Efficiency: 40%
Pressure Ratio: 19.2
Compression Stages: 13
Turbine Stages: 4

Brayton with Intercooling, Reheat and Regeneration

Images removed due to copyright restrictions. Please see Fig. 9-43 and 9-44 in Çengel, Yunus A., and Michael A. Boles. *Thermodynamics: An Engineering Approach*. 5th ed. Boston, MA: McGraw-Hill, 2006. ISBN: 9780072884951.

Gas Turbine Advantages

- Operate at high temperature → Utilize fossil fuel combustion
- Start, stop, turn-down easily → Load following and peaking
- Compact and easy to operate
- Operate at low pressures relative to steam turbines
- Internal combustion does not require heat transfer equipment
- Not as vulnerable to corrosion as steam turbines

Gas Turbine Disadvantages

- Open cycle limits exhaust pressure to ambient pressure
- Exhaust temperature well above ambient
- Efficiency limited by high compression work
- Cannot use dirty fuels (particulate & sulfur damage blades)
- Exhaust temperature well above ambient

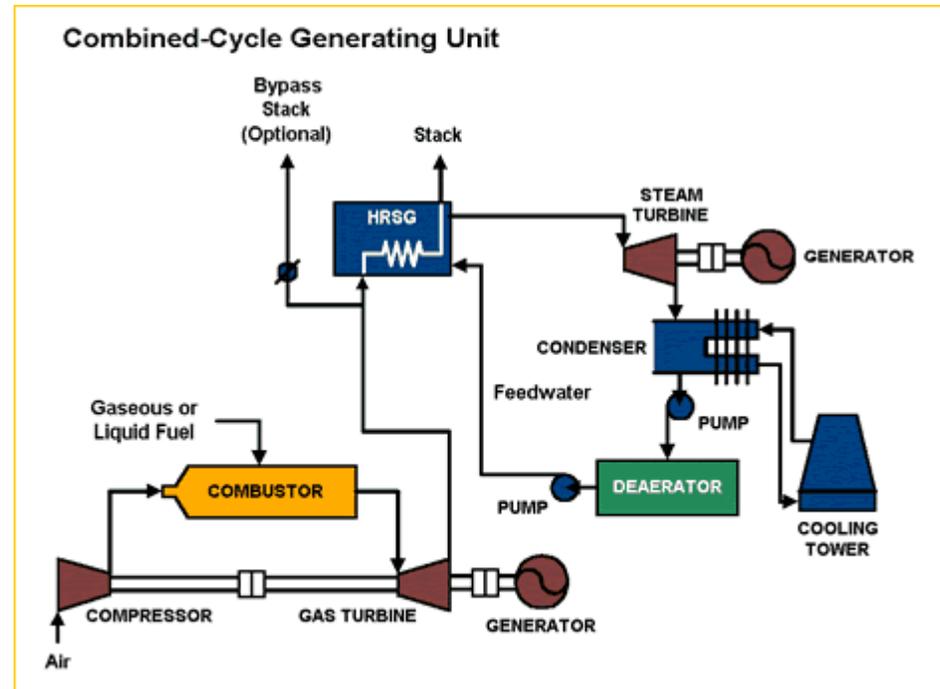
Combined Cycle

Most common combined cycle = Brayton + Rankine

Efficiency $\approx 60\%$

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Figure at right from [U.S. Department of Energy](#).



Sources: MIT Unified Engineering [Lee Langston]; US Department of Energy

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Introduction to Sustainable Energy

Fall 2010

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