## 2.094

# FINITE ELEMENT ANALYSIS OF SOLIDS AND FLUIDS

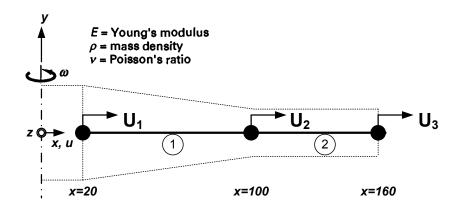
### **SPRING 2008**

### **Homework 2 - Solution**

Assigned: 02/14/2008 Due: 02/21/2008

Instructor: Prof. K. J. Bathe

#### Problem 1 (20 points):



#### In this problem, the global coordinate system is used.

Here, the non-zero strain components are  $[\varepsilon_{\chi\chi}, \varepsilon_{zz}]$ . Hence the stress-strain law represented by the matrix C is

$$\underline{C} = \frac{E}{1 - v^2} \begin{bmatrix} 1 & v \\ v & 1 \end{bmatrix}$$

And

$$\underline{\varepsilon}^T = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{zz} \end{bmatrix} = \begin{bmatrix} \frac{\partial u}{\partial x} & \frac{u}{x} \end{bmatrix}$$

$$\underline{U}^T = \begin{bmatrix} U_1 & U_2 & U_3 \end{bmatrix}$$

The applied force is given by

$$f_x^B(x) = \rho x \omega^2$$
  
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The thickness of each element is given by

$$t^{(1)} = \frac{1}{80}(180 - x)$$
$$t^{(2)} = 1.0$$

For each element, the interpolation functions are

$$\underline{H}^{(1)} = \begin{bmatrix} \frac{1}{80} (100 - x) & -\frac{1}{80} (20 - x) & 0 \end{bmatrix} \text{ for } 20 \le x \le 100$$

$$\underline{H}^{(2)} = \begin{bmatrix} 0 & \frac{1}{60} (160 - x) & -\frac{1}{60} (100 - x) \end{bmatrix} \text{ for } 100 \le x \le 160$$

Then,

$$\underline{B}^{T} = \begin{bmatrix} \frac{\partial H}{\partial x} & \frac{H}{x} \end{bmatrix} 
\underline{B}^{(1)} = \begin{bmatrix} -\frac{1}{80} & \frac{1}{80} & 0 \\ \frac{(100 - x)}{80x} & -\frac{(20 - x)}{80x} & 0 \end{bmatrix} 
\underline{B}^{(2)} = \begin{bmatrix} 0 & -\frac{1}{60} & \frac{1}{60} \\ 0 & \frac{(160 - x)}{60x} & -\frac{(100 - x)}{60x} \end{bmatrix}$$

The stiffness matrices are

$$\underline{K} = \int_{20}^{100} \underline{B}^{(1)^{T}} \underline{C} \ \underline{B}^{(1)} dV^{(1)} + \int_{100}^{160} \underline{B}^{(2)^{T}} \underline{C} \ \underline{B}^{(2)} dV^{(2)} 
= \int_{20}^{100} \underline{B}^{(1)^{T}} \underline{C} \ \underline{B}^{(1)} t^{(1)} 2\pi x dx + \int_{100}^{160} \underline{B}^{(2)^{T}} \underline{C} \ \underline{B}^{(2)} t^{(2)} 2\pi x dx$$

The external force vector is

$$\underline{R} = \int_{20}^{100} \underline{H}^{(1)} \rho x \omega^{2} dV^{(1)} + \int_{100}^{160} \underline{H}^{(2)} \rho x \omega^{2} dV^{(2)} 
= \int_{20}^{100} \underline{H}^{(1)} \rho x \omega^{2} t^{(1)} 2\pi x dx + \int_{100}^{160} \underline{H}^{(2)} \rho x \omega^{2} t^{(2)} 2\pi x dx$$

Therefore we have

$$KU = R$$

After integrations using Matlab,

$$\underline{K} = \frac{E}{1 - \nu^2} \begin{bmatrix} 15.2590 - 10.4704\nu & -4.0988 + 1.0470\nu & 0 \\ -4.0988 + 1.0470\nu & 24.1192 + 2.0943\nu & -13.125 \\ 0 & -13.125 & 14.4885 + 6.2820\nu \end{bmatrix}$$

$$\underline{R} = \rho \omega^2 \begin{bmatrix} 944991 \\ 4521380 \\ 3732212 \end{bmatrix}$$

The same results are obtained using a local coordinate system for each element (like the example 4.5 in the textbook). For your reference, a sample solution using a local coordinate system is attached.

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