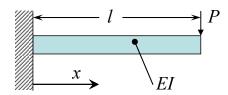
Lecture 8

Energy Methods in Elasticity

Problem 8-1: Consider an elastic cantilever beam loaded at its tip.



- a) Specify the boundary conditions.
- b) Derive the load-tip displacement relation using four methods presented in class.

Method I Solving uncoupled problems

Method II Solving coupled problem (direct integration)

Method III Castigliano Theorem

Method IV Ritz Method

Problem 8-1 Solution:

(a) Boundary conditions

$$w(0) = 0$$

$$w'(0) = 0$$

- (b) Derive the load-tip displacement relation using four methods
 - 1. Solving second order uncoupled problems

We will use

moment-curvature relationship curvature-displacement relationship

 $M = EI\kappa$

 $\kappa = -w$ "

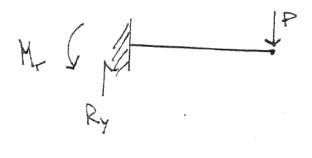
to solve the problem.

These relationships combine to give:

$$-EIw" = M$$

We need the moment.

Find reactions



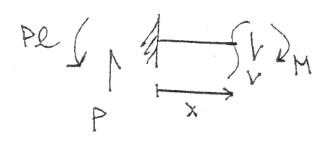
$$\sum F_y = R_y - P = 0$$

$$R_y = P$$

$$\sum M_r = -(M_r - Pl) = 0$$

$$M_r = Pl$$

Find moment in beam base on force balance and moment balance



$$\sum F_{y} = P - V = 0$$

$$V = P$$

$$\sum M_{0} = -(Pl - Vx - M) = 0$$

$$M = -Pl + Vx$$

Combing the above results from force balance and moment balance

$$M = -Pl + Px$$

Integrate moment once

$$-EIw" = M$$

$$\int w' dx = -\int -\frac{1}{EI} (Pl - Px)$$

$$w' = \frac{1}{EI} \left(Plx - \frac{Px^2}{2} \right) + C_1$$

Use B.C.

$$w'(0) = 0$$
$$\Rightarrow C_1 = 0$$

Integrate once

$$w = \int \frac{1}{EI} \left(Plx - \frac{Px^2}{2} \right)$$

$$w = \frac{1}{EI} \left(\frac{Plx^2}{2} - \frac{Px^3}{6} \right) + C_2$$

Use B.C.

$$w(0) = 0$$
$$\Rightarrow C_2 = 0$$

$$w(x) = \frac{1}{EI} \left(\frac{Plx^2}{2} - \frac{Px^3}{6} \right)$$
$$w(l) = \frac{Pl^3}{3EI}$$

2. Solve coupled problems using 4th order direct integration

We use the 4th order differentiated equation:

$$EIw^{IV} = q = 0$$

And we need four boundary conditions:

$$w(0) = 0$$

 $w'(0) = 0$
 $M(0) = -Pl = -EIw''$
 $M(l) = 0 = -EIw''$

Integrate and use boundary conditions to determine the constants

$$\int EIw^{\mathrm{IV}}dx = 0$$

$$EIw''' = C_1$$

$$EIw" = C_1x + C_2$$

Use B.C.

$$M(0) = -Pl$$

$$M(l) = 0$$

We get

$$C_1 = -P$$

$$C_2 = Pl$$

Then

$$EIw" = -Px + Pl$$

$$EIw' = -\frac{Px^2}{2} + Plx + C_3$$

Use B.C.

$$w'(0) = 0$$

We get

$$C_3 = 0$$

Then

$$EIw = -\frac{Px^3}{6} + \frac{Plx^2}{2} + C_4$$

Use B.C.

$$w(0) = 0$$

We get

$$C_4 = 0$$

Finally

$$w(x) = \frac{1}{EI} \left(\frac{Plx^2}{2} - \frac{Px^3}{6} \right)$$
$$w(l) = \frac{Pl^3}{3EI}$$

3. Castigliano Theorem

Find moment in beam

$$M = -Pl + Px$$

Calculate strain energy

$$U = \frac{1}{2} \int \frac{M^2}{EI} dx$$
$$= \frac{1}{2} \int \frac{(Px - Pl)^2}{EI} dx$$
$$= \frac{P^2 l^3}{6EI}$$

Use Castigliano Theorem

$$w(l) = \frac{\partial U}{\partial P}$$
$$w(l) = \frac{Pl^3}{3EI}$$

4. Ritz method

Assume shape function

$$w(x) = C_1 x^3 + C_2 x^2 + C_3 x + C_4$$

Use boundary conditions

$$w(0) = 0$$
$$w'(0) = 0$$

We get

$$C_3 = C_4 = 0$$

$$w(x) = C_1 x^3 + C_2 x^2$$

Also

$$w(l) = C_1 l^3 + C_2 l^2 = w_o$$

Find total potential energy

$$\Pi = \frac{1}{2} \int_{0}^{l} EI(w'')^{2} dx - Pw_{o}$$

$$= \frac{1}{2} \int_{0}^{l} EI(6C_{1}x + 2C_{2})^{2} dx - Pw_{o}$$

$$= \frac{EI}{2} \left(12C_{1}^{2}l^{3} + 12C_{1}C_{2}l^{2} + 4C_{2}^{2}l \right) - Pw_{o}$$

 $\text{Minimize}\,\Pi$

$$\frac{\partial \Pi}{\partial C_1} = 0 \text{ and } \frac{\partial \Pi}{\partial C_2} = 0$$

$$\frac{\partial \Pi}{\partial C_1} = \frac{EI}{2} \left(24C_1l^3 + 12C_2l^2 \right) - Pl^3 = 0$$

$$\frac{\partial \Pi}{\partial C_2} = \frac{EI}{2} \left(12C_1l^2 + 8C_2l \right) - Pl^2 = 0$$

Solve for C_1 and C_2 using the above two equations

$$C_1 = -\frac{P}{6EI}$$
$$C_2 = \frac{Pl}{2EI}$$

Finally

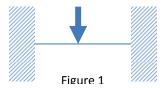
$$w(x) = \frac{1}{EI} \left(\frac{Plx^2}{2} - \frac{Px^3}{6} \right)$$
$$w(l) = \frac{Pl^3}{3EI}$$

Note: we obtain the exact solution with

$$w(x) \sim O(3)$$

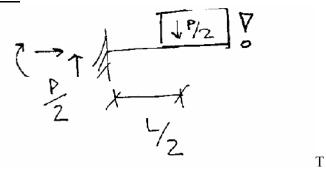
If we were to assume shape function of lower order, we can only obtain an approximate solution.

Problem 8-2: Consider a beam of length L and bending rigidity EI which is fully clamped on both ends shown in Figure 1. The beam is subjected to a point force P at the midspan. Solve the problem (find the expression between the load and the deflection under the load) using:



- The direct integration of beam equation with the suitable boundary conditions (exact solution).
- The Ritz Method (approximate solution).
- Compare the results and calculate the relative error of the Ritz Method.

Problem 8-2 Solution:



Solve for half the beam. Boundary conditions are:

$$w(0) = 0 \tag{1}$$

$$w'(0) = 0 \tag{2}$$

$$w'(\frac{l}{2}) = 0 \tag{3}$$

$$w'(\frac{l}{2}) = 0$$
 (3)
 $V = EIw'''(0) = -\frac{P}{2}$

Direct integration

We use the 4th order differentiated equation:

$$EIw^{IV} = q$$

where q = 0

Integrate once

$$w''' = C_1$$

Use B.C. (4), we have

$$w'''(0) = -\frac{P}{2EI} = C_1$$
$$\Rightarrow w''' = -\frac{P}{2EI}$$

By integration

$$w'' = -\frac{P}{2EI}x + C_2$$

$$w' = -\frac{P}{4EI}x^2 + C_2x + C_3$$

Use B.C. (2), we have

$$w'(0) = C_3 = 0$$
$$\Rightarrow w' = -\frac{P}{4EI}x^2 + C_2x$$

Use B.C. (3),

$$w'\left(\frac{l}{2}\right) = 0$$

$$\Rightarrow C_2 = \frac{Pl}{8EI}$$

$$\Rightarrow w' = -\frac{P}{4EI}x^2 + \frac{Pl}{8EI}x$$

Integrate once

$$w = -\frac{P}{12EI}x^3 + \frac{Pl}{16EI}x^2 + C_4$$

Use B.C. (1),

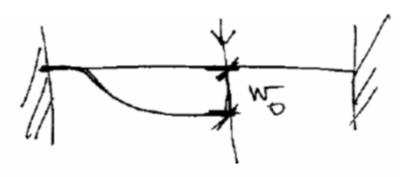
$$w(0) = 0$$
$$\Rightarrow C_4 = 0$$

$$\Rightarrow \boxed{w = -\frac{P}{12EI}x^3 + \frac{Pl}{16EI}x^2, 0 < x < \frac{l}{2}}$$

Mid-span deflection

$$w\left(\frac{l}{2}\right) = \frac{Pl^3}{192EI}$$

b) Ritz method



Assume shape function

$$y(x) = C_1 x^3 + C_2 x^2 + C_3 x + C_4$$

Boundary conditions are:

$$y(0) = 0 \tag{5}$$

$$y'(0) = 0$$
 (6)

$$y'(\frac{l}{2}) = 0 \tag{7}$$

$$y(\frac{l}{2}) = w_0 \tag{8}$$

Use B.C. (5) and (6),

$$C_3 = C_4 = 0$$

$$\Rightarrow y(x) = C_1 x^3 + C_2 x^2$$

Apply B.C. (7) and (8),

$$y'(\frac{l}{2}) = 0$$

$$\Rightarrow 3C_1 \frac{l^2}{4} + 2C_2 \frac{l}{2} = 0$$

$$y(\frac{l}{2}) = w_0$$

$$\Rightarrow C_1 \frac{l^3}{8} + C_2 \frac{l^2}{4} = w_0$$

Combine the above two equations, we can solve for C_1 and C_2

$$C_1 = -\frac{16}{l^3} w_0$$
$$C_2 = \frac{12}{l^2} w_0$$

Calculate $\prod = U - W$

$$y''(x) = \frac{24w_0}{l^2} \left(-\frac{4x}{l} + 1 \right)$$

$$\int_{0}^{l} \left(24w_0 \left(-\frac{4x}{l} + 1 \right) \right)^2 dx = 0.6 \text{ EV } W_0$$

$$U = \frac{1}{2}EI\int_{0}^{l} \left(\frac{24w_{0}}{l^{2}}\left(-\frac{4x}{l}+1\right)\right)^{2} dx = 96EI\frac{w_{0}^{2}}{l^{3}}$$

Minimize \prod

$$\frac{\partial \prod}{\partial w_0} = 96EI \frac{2w_0}{l^3} - P = 0$$

$$w_0 = \frac{Pl^3}{192EI}$$

Finally

$$w = -\frac{P}{12EI}x^3 + \frac{Pl}{16EI}x^2, 0 < x < \frac{l}{2}$$

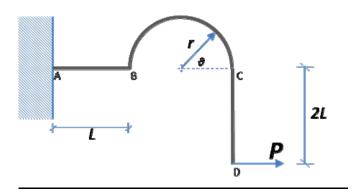
c) Compare Ritz method and exact solution

$$w_0\big|_{\text{exact}} = \frac{Pl^3}{192EI}$$

$$w_0\big|_{\text{Ritz}} = \frac{Pl^3}{192EI}$$

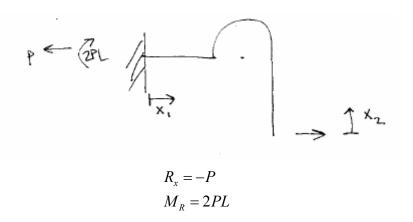
Because we guess the correct order of the shape function, we got the exact solution. %error=0

Problem 8-3: Use Castigliano's Theorem to calculate the horizontal deflection at point D in Figure



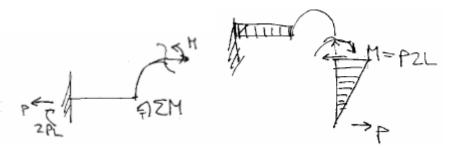
Problem 8-3 Solution:

Find reaction



where R_{x} and M_{R} are reaction force and reaction moment

Find internal moment distribution at each section



Section AB

$$M_1 = 2PL$$

Section BC

$$M_2 = P(2L + r\sin\theta)$$

Section CD

$$M_3 = Px_2$$

Calculate strain energy

$$U = U_1 + U_2 + U_3$$

$$= \frac{1}{2} \int_0^L \frac{(2PL)^2}{EI} dx_1 + \frac{1}{2} \int_0^\pi \frac{(P(2L + r \sin \theta))^2}{EI} r d\theta + \frac{1}{2} \int_0^{2L} \frac{(Px_2)^2}{EI} dx_2$$

$$= \frac{1}{2EI} \left\{ (2PL)^2 L + rP^2 \int_0^\pi \left[4L^2 + 2(2Lr \sin \theta) + r^2 \sin^2 \theta \right] d\theta + \frac{8P^2L^3}{3} \right\}$$

$$= \frac{P^2}{2EI} \left(\frac{20L^3}{3} + 4\pi L^2 r + 8Lr^2 + \frac{\pi r^3}{2} \right)$$

Use Castigliano

$$w_0 = \frac{\partial U}{\partial P} = \frac{2P}{2EI} \left(\frac{20L^3}{3} + 4\pi L^2 r + 8Lr^2 + \frac{\pi r^3}{2} \right)$$

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