

Lecture 9, Thermal Notes, 3.054

Thermal Properties of Foams

- Closed cell foams widely used for thermal insulation
- Only materials with lower conductivity are aerogels (tend to be brittle and weak) and vacuum insulation panels
- Low thermal conductivity of foam arises from:
 - low volume fraction of solid
 - high volume fraction of gas with low λ
 - small cell size suppresses convection and radiation (through repeated absorption and reflection)
- Applications: buildings, refrigerated vehicles, LNE tankers
- Foams also have good thermal shock resistance since coefficient of thermal expansion of foam equals to that of the solid; plus the modulus is much lower ($\epsilon = \alpha\Delta T$ $\sigma = E\alpha\Delta T = \sigma_f$)
⇒ used as heat shields
- Ceramic foams used as firebrick — ceramic has high T
 - foam - low λ - low heat loss
 - low heat capacity - lowers energy to heat furnace to temperature
 - good thermal shock resistance

Thermal conductivity, λ

- Steady state conduction (T constant with time)

Fourier Law: $q = -\lambda \nabla T$

1D $q = -\lambda \frac{dT}{dx}$

q = heat flux [$\text{J}/(\text{m}^2/\text{s})$]

λ = thermal conductivity [W/mK]

∇T = temperature gradient

$$= i \frac{\partial T}{\partial x} + j \frac{\partial T}{\partial y} + k \frac{\partial T}{\partial z}$$

- Non-steady heat conduction (T varies with time t)

$$\frac{\partial T}{\partial \tau} = a \frac{\partial^2 T}{\partial x^2}$$

a = thermal diffusivity = $\frac{\lambda}{\rho C_p}$
[m^2/s]

ρ = density

C_p = specific heat - heat required to

raise the temperature of unit mass by 1°K

ρC_p = volumetric heat capacity [$\text{J}/\text{m}^3\text{K}$]

- Values for λ , a Table 7.1

Table 7.1 Thermal conductivities and diffusivities

Material	Thermal conductivity λ (W/m K)	Thermal diffusivity a (m ² /s)
Copper (solid)	384 ^a	8.8×10^{-5} a
Aluminium (solid)	230 ^a	8.9×10^{-5} a
Alumina (solid)	25.6 ^a	8.2×10^{-6} a
Glass (solid)	1.1 ^a	4.5×10^{-7} a
Polyethylene (solid)	0.35 ^a	1.7×10^{-7} a
Polyurethane (solid)	0.25 ^c	
Polystyrene (solid)	0.15 ^a	1.0×10^{-7} a
Air	0.025 ^a	–
Carbon dioxide	0.016 ^a	–
Trichlorofluoromethane (CCl ₃ F)	0.008 ^a	–
Oak ($\rho^*/\rho_s = 0.40$)	0.150 ^a	–
White pine ($\rho^*/\rho_s = 0.34$)	0.112 ^a	–
Balsa ($\rho^*/\rho_s = 0.09$)	0.055 ^a	–
Cork ($\rho^*/\rho_s = 0.14$)	0.045 ^a	–
Polystyrene foam ($\rho^*/\rho_s = 0.025$)	0.040 ^b	1.1×10^{-6} b
Polyurethane foam ($\rho^*/\rho_s = 0.02$)	0.025 ^b	9.0×10^{-7} b
Polystyrene foam ($\rho^*/\rho_s = 0.029$ – 0.057)	0.029–0.035 ^d	
Polyisocyanurate foam, (CFC-11) ($\rho^* = 32$ kg/m ³)	0.020 ^d	
Phenolic foam, (CFC-11, CFC-113) ($\rho^* = 48$ kg/m ³)	0.017 ^d	
Glass foam ($\rho^*/\rho_s = 0.05$)	0.050 ^d	
Glass wool ($\rho^*/\rho_s = 0.01$)	0.042 ^d	
Mineral fibre ($\rho^*/\rho_s = 4.8$ – 32 kg/m ³)	0.046 ^d	

All values for room temperature.

References

^aHandbook of Chemistry and Physics, 66th edn (1985–6) Chemical Rubber Co. ed. R. C. Weast.

^bPatten, G. A. and Skochdopole, R. E. (1962) *Mod. Plast.*, **39**, 149.

^cSchuetz, M. A. and Glicksman, L. R. (1983) *Proc. SPI 6th International Technical/Marketing Conference*, pp. 332–40.

^dGlicksman, L. R. (1994) Heat transfer in foams, in *Low Density Cellular Plastics* ed. Hilyard, N. C. and Cunningham, A. Chapman and Hall.

Data for thermal conductivity and thermal diffusivity

Thermal diffusivity, a

- Materials with a high value of a rapidly adjust their temperature to that of surroundings, because they conduct heat rapidly in comparison to their volumetric heat capacity; do not require much energy to reach thermal equilibrium

e.g. Cu $a = 112 \times 10^{-6} \text{ m}^2/\text{s}$

nylon $a = 0.09 \times 10^{-6} \text{ m}^2/\text{s}$

wood $a = 0.082 \times 10^{-6} \text{ m}^2/\text{s}$

Thermal conductivity of a foam, λ^* .

λ^* — contributions from — conduction through solid, λ_s^*

— conduction through gas, λ_g^*

— convection within cells, λ_c^*

— radiation through cell walls and across voids, λ_r^*

$$\lambda^* = \lambda_s^* + \lambda_g^* + \lambda_c^* + \lambda_r^*$$

- Conduction through solid: $\lambda_s^* = \eta \lambda_s (\rho^*/\rho_s)$ $\eta = \text{efficiency factor} \sim 2/3$
- Conduction through gas: $\lambda_g^* = \lambda_g (1 - \rho^*/\rho_s)$

For example, 2.5% dense closed-cell polystyrene foam:

$$\lambda^* = 0.040 \text{ W/mK}; \lambda_s^* = 0.15 \text{ W/mK}; \lambda_g^* = 0.025 \text{ W/mK (air)}$$

$$\begin{aligned} \lambda_s^* + \lambda_g^* &= 2/3 (0.15)(0.025) + (0.025)(0.975) \\ &= 0.003 + 0.024 \\ &= 0.027 \text{ W/mK} \end{aligned}$$

- Most of conductivity comes from conduction through gas
- Foams for isolation blown with low λ_g gases
- Problem with aging — low λ_g gases diffuse out of foam over time, air diffuses in; $\lambda_g^* \uparrow$

Convection within the cell



- Gas rises and falls due to density changes with temperature
- Density changes — buoyancy forces
- Also have viscous forces from drag of gas as it moves past cell wall

Convection is important when Rayleigh number > 1000

$$R_a = \frac{\rho g \beta \Delta T_c l^3}{\mu a}$$

ρ = density of gas

g = grav. acceleration

β = volume expansion

for a gas = $1/T$ (isobaric)

ΔT_c = temp. diff. across the cell

l = cell size

μ = dynamic viscosity of gas

a = thermal diffusion

Convection

For $R_a = 1000$ **air** $p = p_{\text{atm}}$ $T = \text{room temp}$ $\beta = 1/T = 1/300$ ($^{\circ}\text{K}^{-1}$).
 $\Delta T_c = 1^{\circ}\text{K}$ $\mu_{\text{air}} = 2 \times 10^{-5} \text{ Pa}\cdot\text{s}$ $\rho_{\text{air}} = 1.2 \text{ kg/m}^3$
 $a_{\text{air}} = 2.0 \times 10^{-5} \text{ m}^2/\text{s}$
 $\Rightarrow l = 20 \text{ mm}$

- Convection important if cell size $> 20 \text{ mm}$
- Most foams: cell size $< 1 \text{ mm} \Rightarrow$ convection negligible

Radiation

- Heat flux passing by radiation, q_r^0 , from surface at temperature T_1 , to one at a lower temperature T_0 , with a vacuum between them, is:

$$q_r^0 = \beta_1 \sigma (T_1^4 - T_0^4) \quad \text{Stefan's law}$$

$$\sigma = \text{Stefan's constant} = 5.67 \times 10^{-8} \text{ W/m}^2 \text{ K}^4$$

$\beta_1 = \text{constant} (< 1)$ describing emissivity of the surfaces

(emitted radiant flux per unit area of sample relative to black body radiator at same temperature and conditions; black body absorbs all energy; black body emissivity = 1)

Radiation

- If put foam between two surfaces, heat flux is reduced, since radiation is absorbed by the solid and reflected by cell walls
- Attenuation $q_r = q_r^0 \exp(-K^* t^*)$ Beer's law
 K^* = extinction coefficient for foam
 t^* = thickness of foam

- For optically thin walls and struts ($t < 10\mu m$) (transparent to radiation)

$$K^* = (\rho^*/\rho_s) K_s$$

- Heat flux by radiation then:

$$q_r = \lambda_r^* \frac{dT}{dx}$$

$$q_r = \beta_1 \sigma (T_1^4 - T_0^4) \exp[-(\rho^*/\rho_s) K_s t^*] = \lambda_r^* \frac{dT}{dx}$$

- Obtain λ_r using some approximations

Approximations:

$$\frac{dT}{dx} \approx \frac{T_1 - T_0}{t^*} = \frac{\Delta T}{t^*}$$

$$T_1^4 - T_0^4 \approx 4 \Delta T \bar{T}^3 \quad \bar{T} = \left(\frac{T_1 + T_0}{2} \right)$$

$$q_r = \beta_1 \sigma 4 \Delta T \bar{T}^3 \exp [-(\rho^*/\rho_s) K_s t^*] = \lambda_r^* \frac{\Delta T}{dx}$$

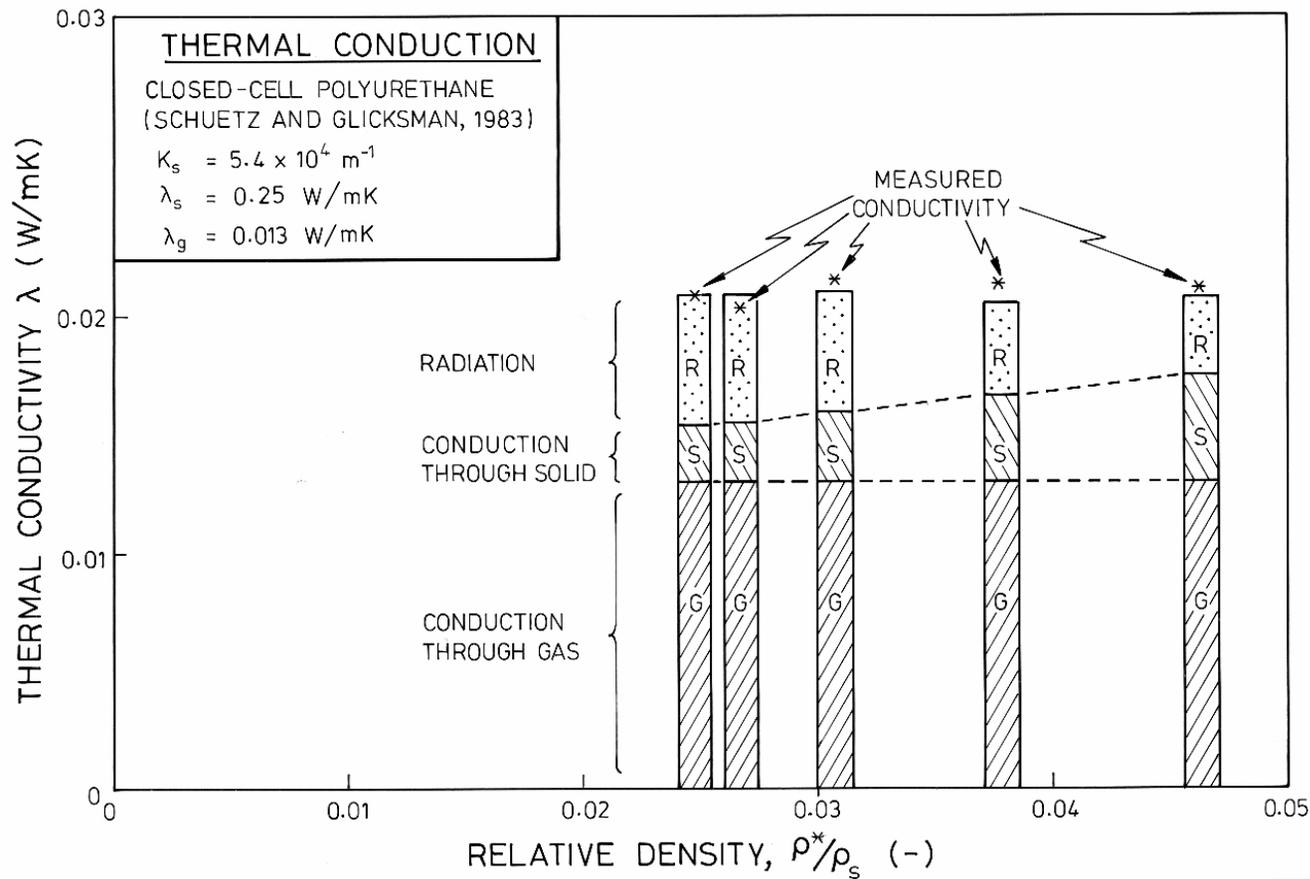
$$\lambda_r^* = 4 \beta_1 \sigma \bar{T}^3 t^* \exp [-(\rho^*/\rho_s) K_s t^*]$$

as $\rho^*/\rho_s \downarrow$ $\lambda_r^* \uparrow$

Thermal conductivity

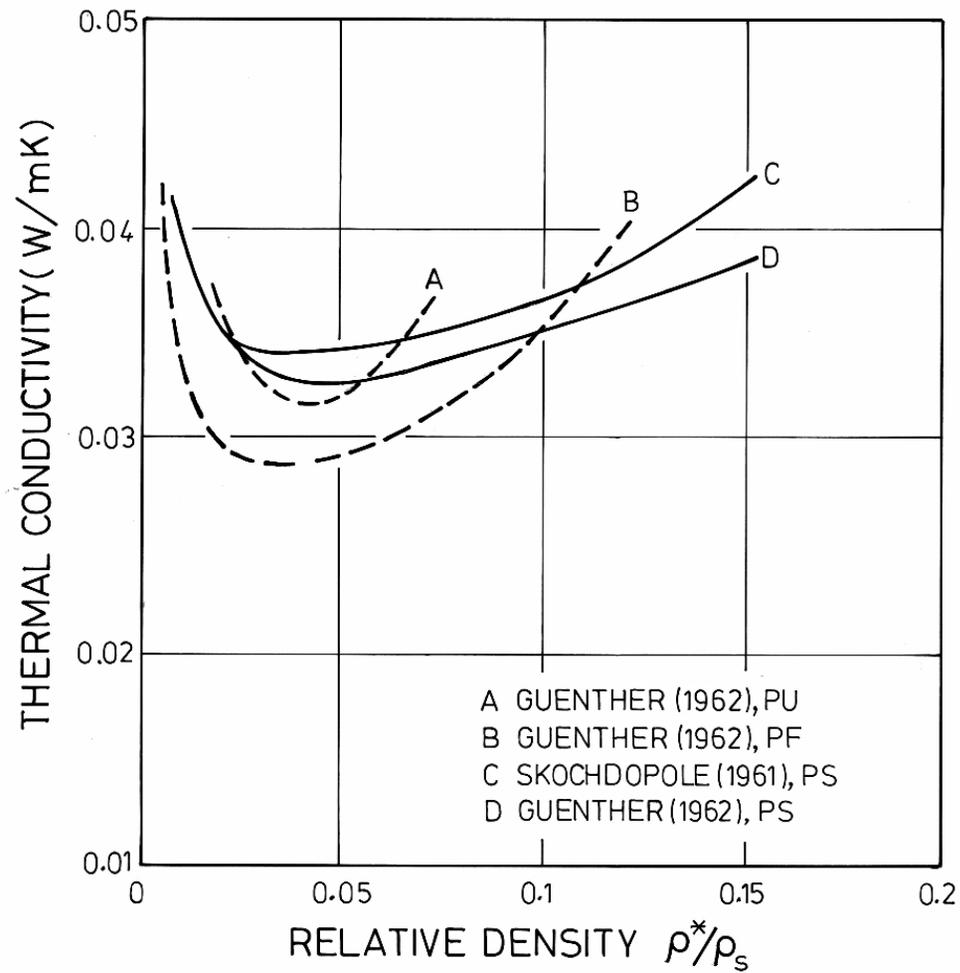
- Relative contributions of λ_s^* , λ_g^* , λ_r^* shown in Fig. 7.1
 - largest contribution λ_g^*
- λ^* plotted against relative density Fig. 7.2
 - minimum between ρ^*/ρ_s of 0.03 and 0.07
 - at which point λ^* only slightly larger than λ_s^*
 - at low ρ^*/ρ_s , λ^* increases - increasing transparency to radiation (also, walls may rupture)
 - tradeoff: as ρ^*/ρ_s goes down, λ_s^* goes down, but λ_r^* goes up

Thermal Conductivity



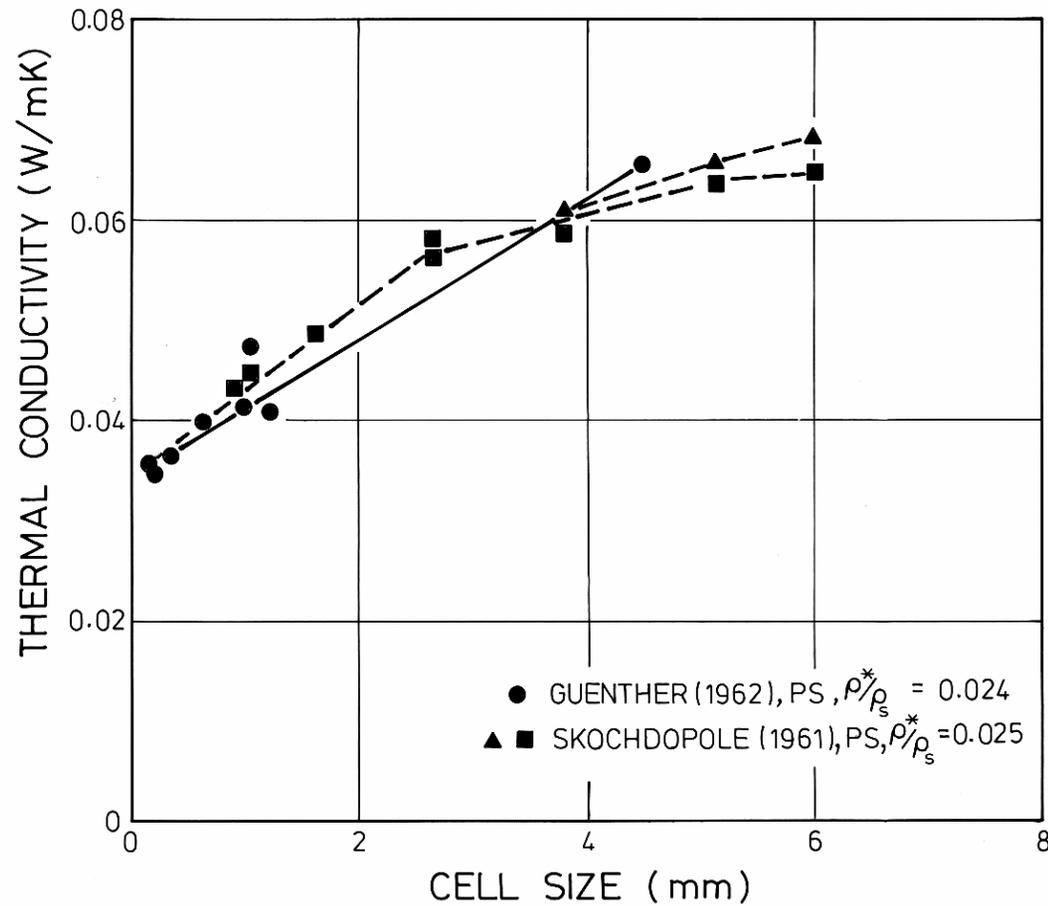
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Cond. Vs. Relative Density



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Cond. vs. Cell Size



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λ^* plotted against cell size Fig. 7.3

- λ^* increases with cell size
- Radiation reflected less often

Note: aerogels

- Pore size $< 100nm$
- Mean free path of air at ambient pressure = 68 nm
→ average distance molecules move before collision with another molecule
- Aerogels — pore size $<$ mean free path of air — reduced conduction through gas

Specific heat C_p

- Specific heat — energy required to raise temperature of unit mass by unit temperature

$$C_p^* = C_{ps} \quad [\text{J/kg} \cdot \text{K}]$$

Thermal expansion coefficient

$$\alpha^* = \alpha_s \quad (\text{consider foam as framework})$$

(but if closed-cell foam cooled dramatically — gas can freeze, collapsing the cells; or if heated — gas expands, increasing the internal pressure and strains)

Thermal shock resistance

- If material subjected to sudden change in surface temperature - induces thermal stresses at surface, plus cracking and spalling
- Consider material at T_1 dropped into water at T_2 ($T_1 > T_2$)
 - Surface temperature drops to T_2 , contracting surface layers
 - Thermal strain $\epsilon_T = \alpha \Delta T$
- If surface bonded to underlying block of material - constrained to original dimensions

$$\sigma = \frac{E \alpha \Delta T}{1 - \nu} \quad \text{in the surface}$$

- Cracking/spalling when $\sigma = \sigma_f$

$$\Delta T = \sigma_f \frac{1 - \nu}{E \alpha} = \text{critical } \Delta T \text{ to just cause cracking}$$

- For foam: (open cells)

$$\Delta T_c^* = \frac{0.2 \sigma_{fs} (\rho^*/\rho_s)^{3/2} (1 - \nu^*)}{E_s (\rho^*/\rho_s)^2 \alpha_s} = \frac{0.2}{(\rho^*/\rho_s)^{1/2}} \frac{\sigma_{fs} (1 - \nu)}{E_s \alpha_s} = \frac{0.2}{(\rho^*/\rho_s)^{1/2}} \Delta T_{cs}$$

- As foam density goes down, ΔT_c^* goes up firebrick - porous ceramic

Case study: optimization of foam density for thermal insulation

- There is an optimal foam density for a given thermal insulation problem
- Already saw λ^* has a minimum as a $f(\rho^*/\rho_s)$
- Typically, have a constraint on the foam thickness, t^* , $t^* = \text{constant}$

$$\lambda^* = \frac{2}{3}(\rho^*/\rho_s)\lambda_s + (1 - \rho^*/\rho_s)\lambda_g + 4\beta_1\sigma\bar{T}^3 t^* \exp[-K_s(\rho^*/\rho_s)t^*]$$

- What is optimum ρ^*/ρ_s for a given t^* ?

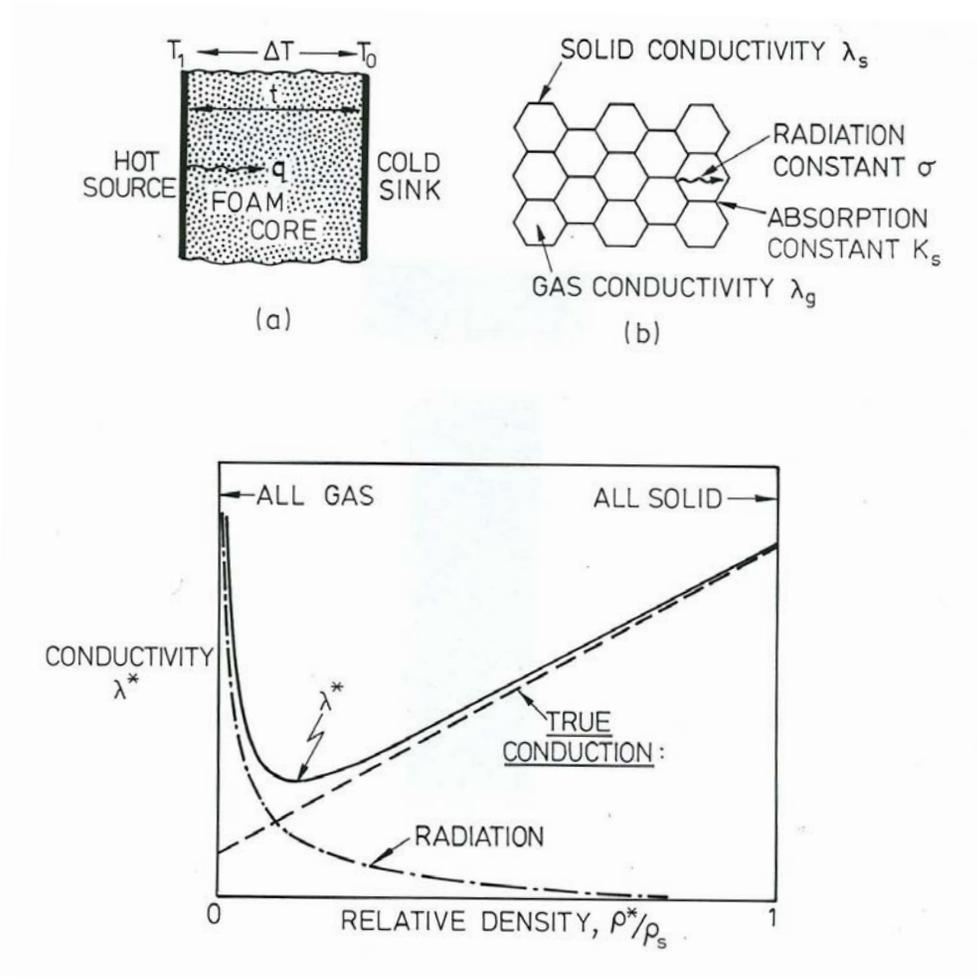
$$\frac{d\lambda^*}{d(\rho^*/\rho_s)} = 0 \Rightarrow (\rho^*/\rho_s)_{\text{opt}} = \frac{1}{K_s t^*} \ln \left[\frac{4K_s\beta_1\sigma\bar{T}^3 t^{*2}}{\frac{2}{3}\lambda_s - \lambda_g} \right]$$

- As given thickness t^* increases, $(\rho^*/\rho_s)_{\text{opt}}$ decreases
- As \bar{T} increases, $(\rho^*/\rho_s)_{\text{opt}}$ increases

e.g. coffee cup $t^* = 3\text{mm}$ $(\rho^*/\rho_s)_{\text{opt}} = 0.08$
refrigerator $t^* = 50\text{mm}$ $(\rho^*/\rho_s)_{\text{opt}} = 0.02$

(see PP slide Table 7.3 for data used in calculations)

Case Study: Optimization of Relative Density



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Case Study: Optimum Relative Density

Table 7.3 Data for optimization case study

Extinction coefficient of solid polymer, K_s	$5.67 \times 10^4 \text{ m}^{-1}$
Emissivity factor, β_1	0.5
Conductivity of solid polymer, λ_s	0.22 W/m K
Conductivity of gas, λ_g	0.02 W/m K
Mean temperature, \bar{T}	300°K
Stefan's constant, σ	$5.67 \times 10^{-8} \text{ W/m}^2\text{K}^4$

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Case study: insulation for refrigerators

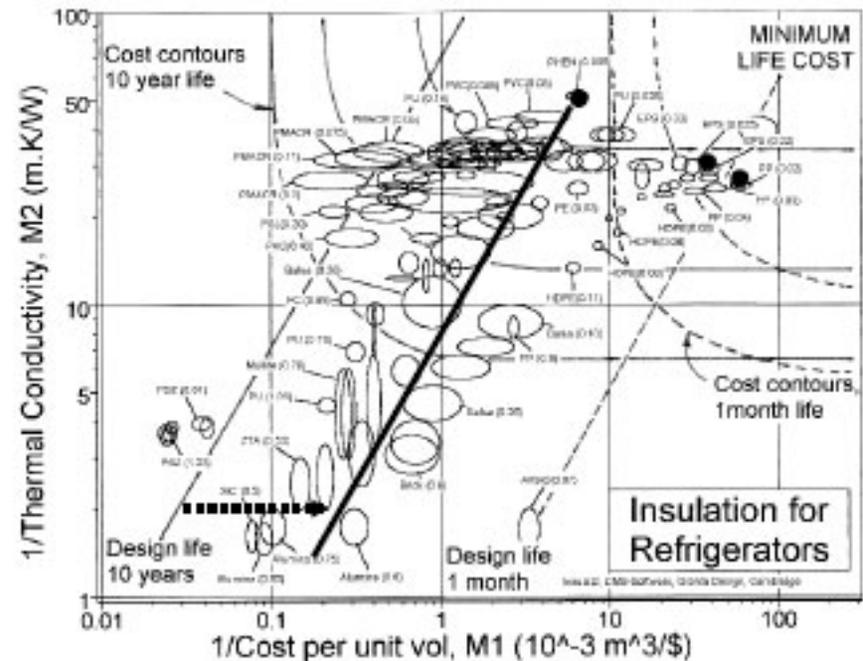
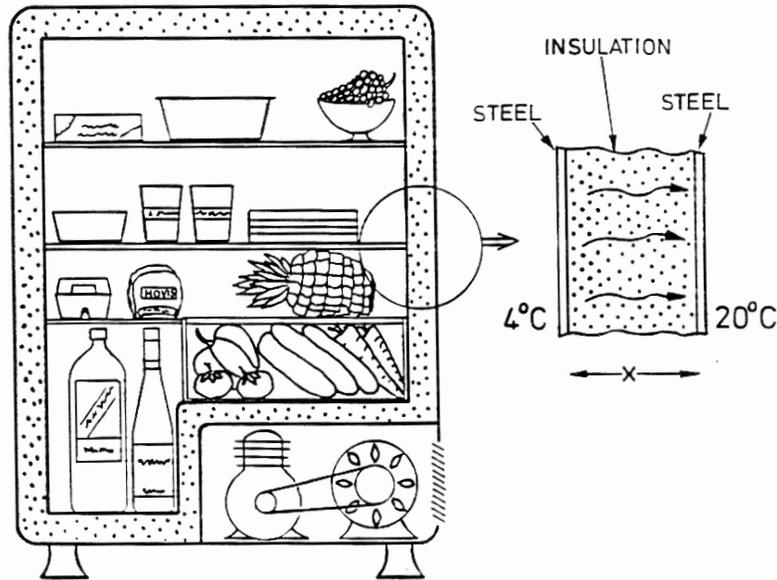
- Insulation reduces energy cost, but has a cost itself
- Total cost is the cost of insulation plus the cost of energy lost by heat transfer through walls
- Objective function: minimize total cost
- given: x =thickness of insulation C_M =cost of insulation/mass
 ΔT =temp. diff. across insulation C_E =cost of energy / joule
 t_l =design life of refrigerator C_T =total cost/area

$$C_T = x \rho^* C_M + \lambda \frac{\Delta T}{x} t_l C_E \quad (\text{heat flux } q = \lambda \frac{\Delta T}{x} \frac{\text{J}}{\text{m}^2\text{s}})$$

$$\text{Define: } M_1 = \frac{1}{\rho^* C_M} \quad M_2 = \frac{1}{\lambda}$$

$$\frac{C_T}{x} = \frac{1}{M_1} + \left[\frac{\Delta T}{x^2} t_l C_E \right] \frac{1}{M_2}$$

Case Study: Insulation for Refrigerators



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