

1.050 Engineering Mechanics

Lecture 14:

Strength models for beams (II/II)

M-N coupling

Convexity of strength domain

1.050 – Content overview

I. Dimensional analysis

1. On monsters, mice and mushrooms
2. Similarity relations: Important engineering tools

Lectures 1-3
Sept.

II. Stresses and strength

2. Stresses and equilibrium
3. Strength models (how to design structures, foundations.. against mechanical failure)

Lectures 4-15
Sept./Oct.

III. Deformation and strain

4. How strain gages work?
5. How to measure deformation in a 3D structure/material?

Lectures 16-19
Oct.

IV. Elasticity

5. Elasticity model – link stresses and deformation
6. Variational methods in elasticity

Lectures 20-31
Nov.

V. How things fail – and how to avoid it

7. Elastic instabilities
8. Plasticity (permanent deformation)
9. Fracture mechanics

Lectures 32-37
Dec.

1.050 – Content overview

I. Dimensional analysis

II. Stresses and strength

...

Lecture 8: Beam stress model

Lecture 9: Beam model II and summary

Lecture 10: Strength models: Introduction (1D)

Lecture 11: Mohr circle – strength criteria 3D

Lecture 12: Application – soil mechanics: How to build sandcastles

Lecture 13: Strength criterion in beams (I/II)

Lecture 14: Strength criterion in beams (II/II) – convexity of strength domain

Lecture 15: Closure strength models & review for quiz

III. Deformation and strain

IV. Elasticity

V. How things fail – and how to avoid it

Quiz I

- **Wednesday, October 17 in class**
- Please be on time
- Covers first 15 lectures
- Open book

- **Preparation:**
 - Lecture material, PSs, recitation
 - Old quizzes (posted) instead of PS this week
 - Alberto will work through one example (nanoindentation) in recitation
 - Study old quizzes before recitation this week

Strength models

- Equilibrium conditions “only” specify statically admissible stress field, without worrying about if the stresses can actually be sustained by the material – **S.A.**
From EQ condition for a REV we can integrate up (upscale) to the structural scale

Examples: Many integrations in homework and in class; Hoover dam etc.

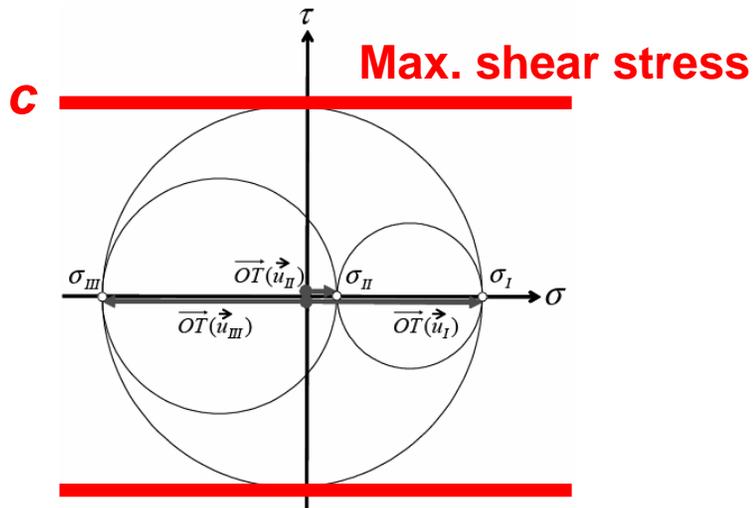
- Strength compatibility adds the condition that in addition to S.A., the stress field must be compatible with the strength capacity of the material – **S.C.**
In other words, at no point in the domain can the stress vector exceed the strength capacity of the material

Examples: Sand pile, foundation etc. – Mohr circle

Strength models

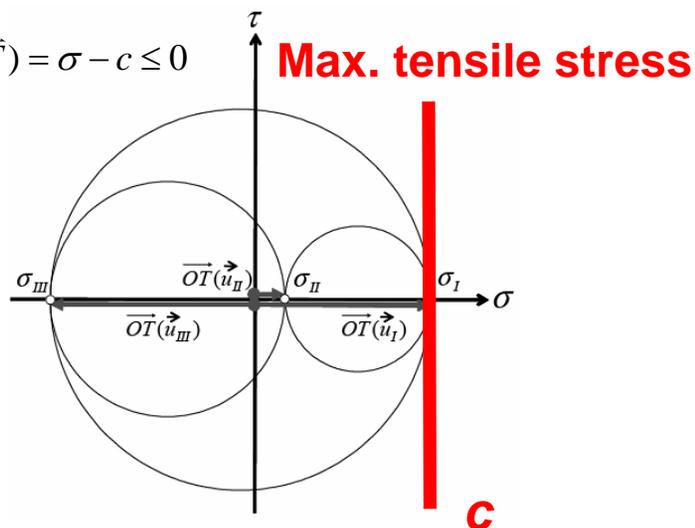
$$\text{Tresca: } \forall \vec{n}; f(\vec{T}) = |\tau| - c \leq 0$$

cohesion, $2c = \sigma_0$



Tresca criterion

$$\forall \vec{n}: f(\vec{T}) = \sigma - c \leq 0$$

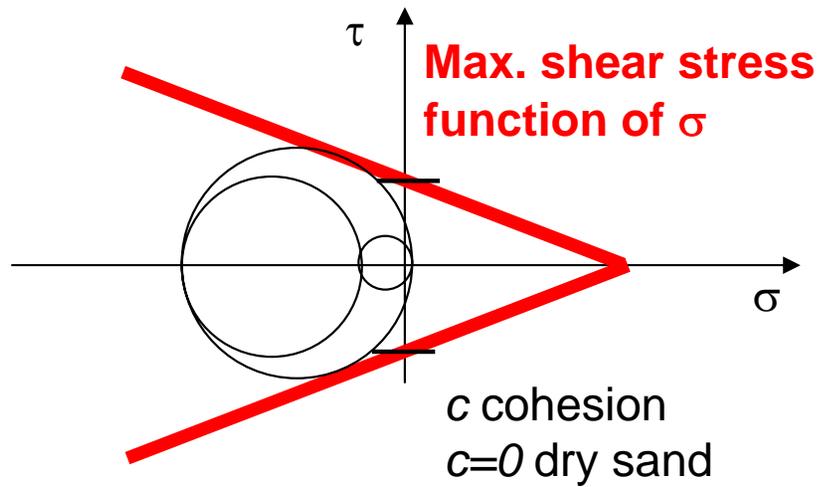


Tension cutoff criterion

Strength models

$$\text{Mohr-Coulomb: } \forall \vec{n}; f(\vec{T}) = |\tau| + \sigma \tan \varphi - c \leq 0$$

μ



Mohr-Coulomb

Review: Beam models

Beam model: Special case of the general continuum model

Special geometry – highly distorted system (much longer than wide)

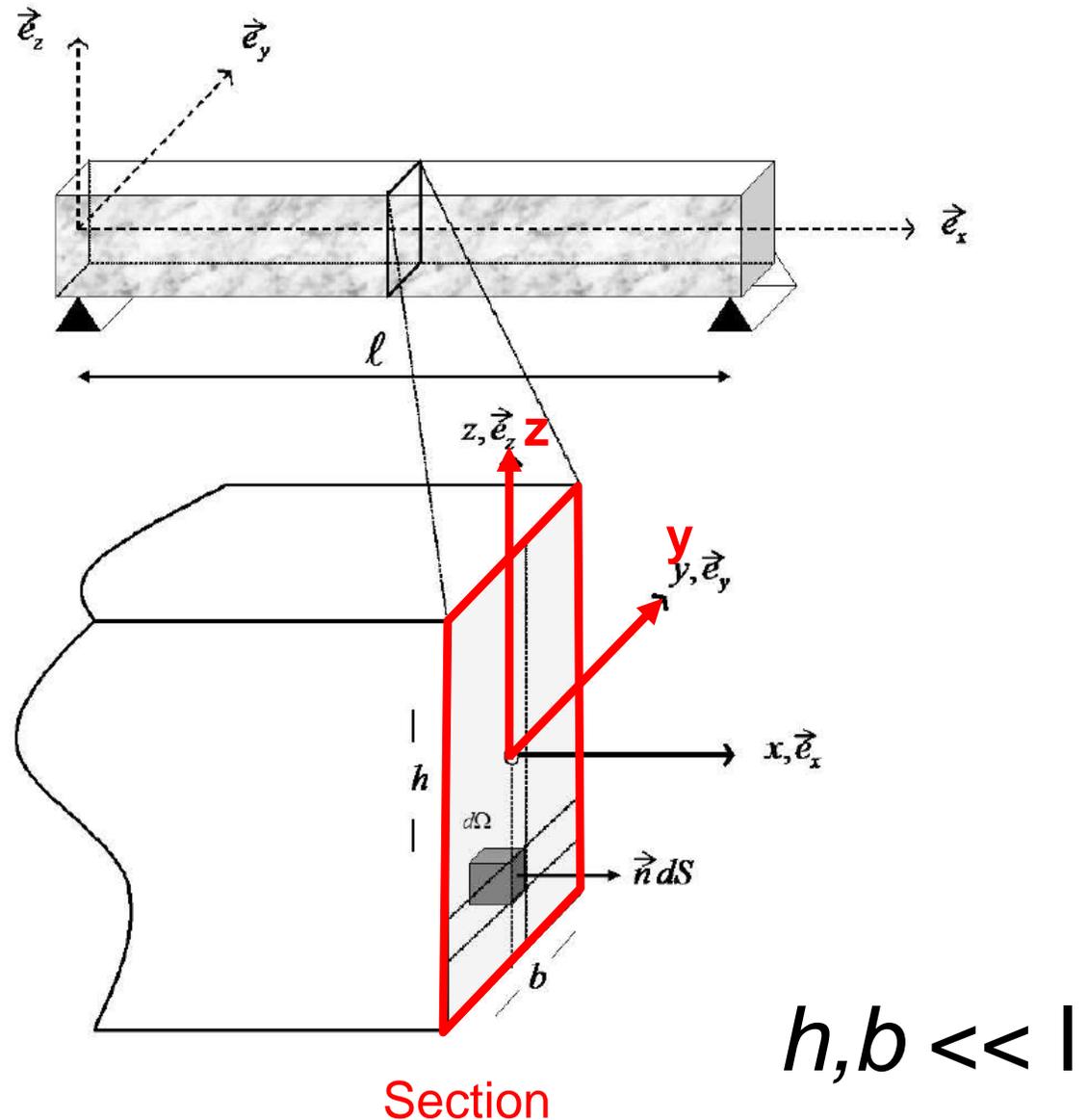
Special form of stress tensor:

$$\sigma_{ij} = \begin{pmatrix} \sigma_{xx} & & \text{sym} \\ \sigma_{yx} & \sim 0 & \\ \sigma_{zx} & \sim 0 & \sim 0 \end{pmatrix}$$

$$\underline{\underline{\sigma}} = \underline{\underline{\sigma}}(x; y, z)$$

For fixed x (section choice):

$$\underline{\underline{\sigma}} = \underline{\underline{\sigma}}(y, z)$$



Link between section quantities and section stress field

- Section force and moment distribution is due to a particular stress tensor distribution in the section

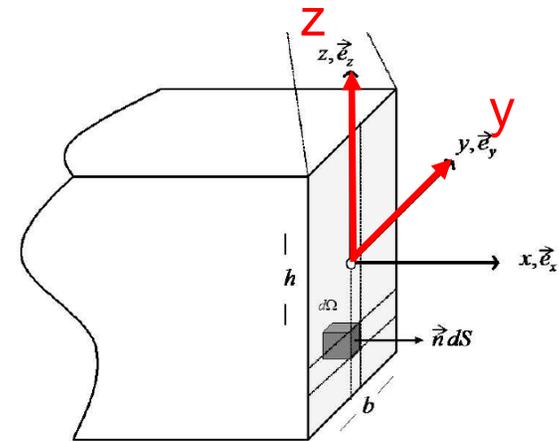
Reduction formulas

$$dS = dzdy$$

$$N_x = \int_S \sigma_{xx}(y, z) dS$$

$$Q_y = \int_S \sigma_{yx}(y, z) dS$$

$$Q_z = \int_S \sigma_{zx}(y, z) dS$$



y, z : C.S. in section

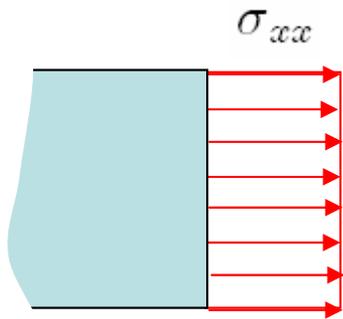
$$\begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix} = \int_S \begin{pmatrix} 0 \\ y \\ z \end{pmatrix} \times \begin{pmatrix} \sigma_{xx} \\ \sigma_{yx} \\ \sigma_{zx} \end{pmatrix} dS = \begin{pmatrix} \int_S [y\sigma_{zx} - z\sigma_{yx}] dS \\ \int_S z\sigma_{xx} dS \\ - \int_S y\sigma_{xx} dS \end{pmatrix}$$

Torsion
 Bending

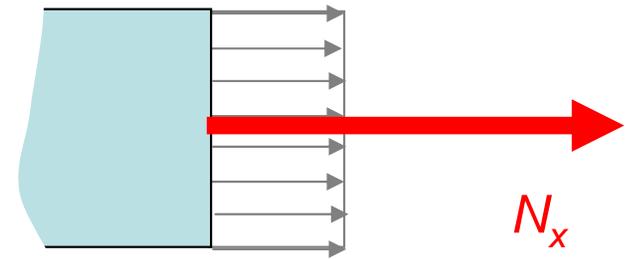
$$\vec{x}_S = y\vec{e}_y + z\vec{e}_z$$

Example

Stress distribution in section



Equivalent normal force

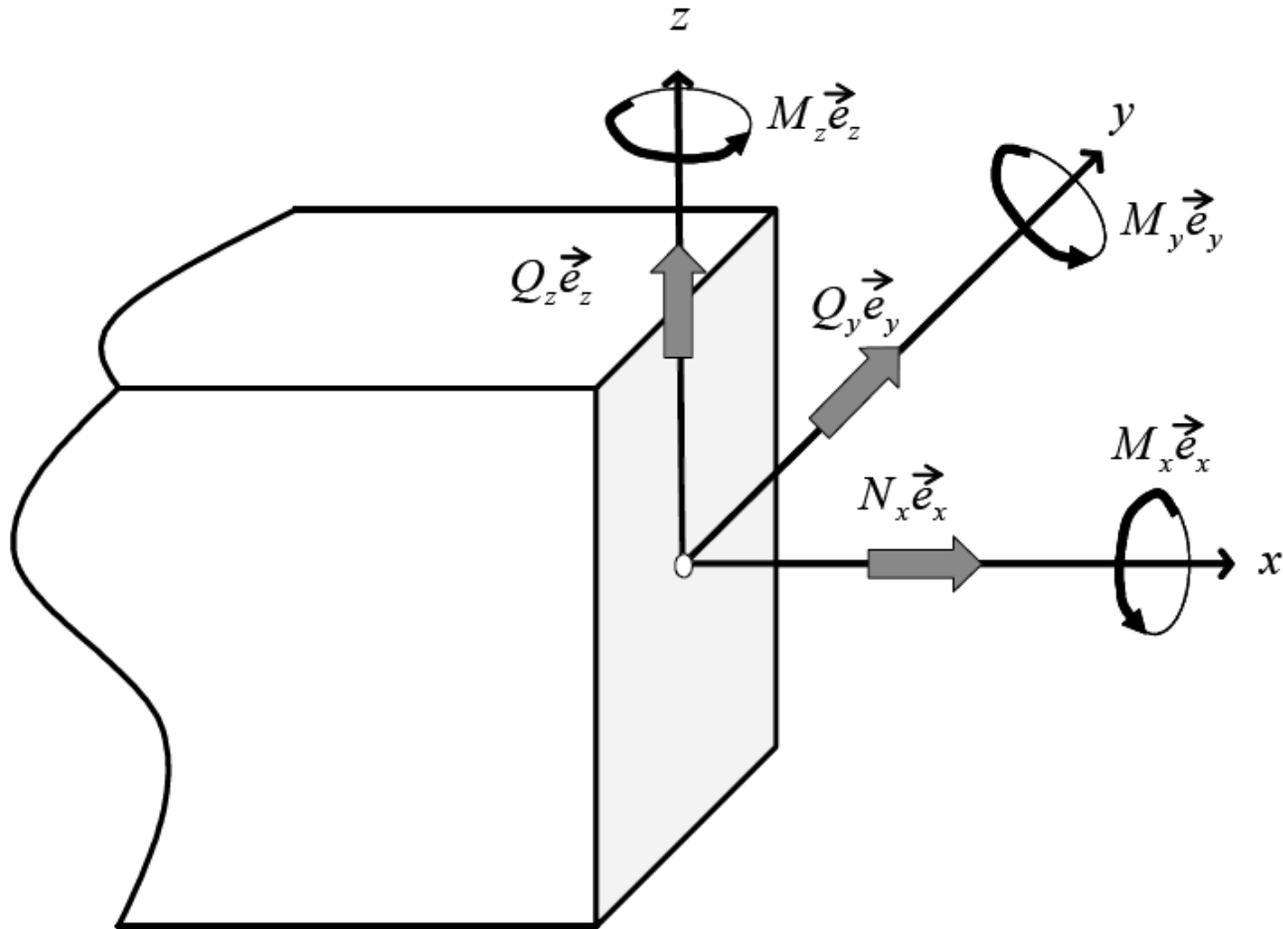


$$N_x = \int_S \sigma_{xx}(y, z) dS$$

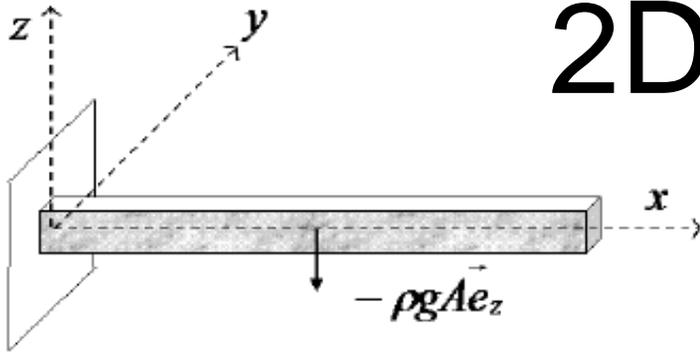
Review: Beam models

	Continuum model	Beam model
Differential element	$d\Omega$	dx
Equilibrium condition	$\text{in } \Omega : \begin{cases} \vec{T}(\vec{n}) = \boldsymbol{\sigma} \cdot \vec{n} \\ \text{div } \boldsymbol{\sigma} + \rho(\vec{g} - \vec{a}) = 0 \\ \sigma_{ij} = \sigma_{ji} \end{cases}$ <div style="text-align: right; margin-right: 20px;">0</div> $\text{on } \partial\Omega; \vec{T}^d = \vec{T}(\vec{n})$	$\forall x \in [0, \ell]; \frac{d\vec{F}_S}{dx} + \vec{f}^{ext} = 0$ $\forall x \in [0, \ell]; \frac{d\vec{M}_S}{dx} + \vec{e}_x \times \vec{F}_S = 0$ $x = 0, \ell : \begin{cases} \vec{F}^d = \vec{F}_S(x) \\ \vec{M}^d = \vec{M}_S(x) \end{cases}$ $\vec{f}^{ext} = f_x \vec{e}_x + f_y \vec{e}_y + f_z \vec{e}_z \quad \text{Line force density}$ $[\vec{f}^{ext}] = [F] L^{-1}$
Simplification	<p>Hydrostatics (fluid):</p> $\text{in } \Omega : \begin{cases} \vec{T}(\vec{n}) = -p\vec{n} \\ \boldsymbol{\sigma} = -p\mathbf{1} \\ -\text{grad } p + \rho\vec{g} = 0 \end{cases}$ <p>+BCs</p> $-\frac{\partial p}{\partial x} = 0; \quad -\frac{\partial p}{\partial y} = 0; \quad -\frac{\partial p}{\partial z} - \rho g = 0$	<p>2D:</p> $\frac{dN_x}{dx} + f_x = 0 \quad \frac{dQ_z}{dx} + f_z = 0$ $\frac{dM_y}{dx} - Q_z = 0$ <p>+BCs</p> 

Beam models: Moments



2D Example



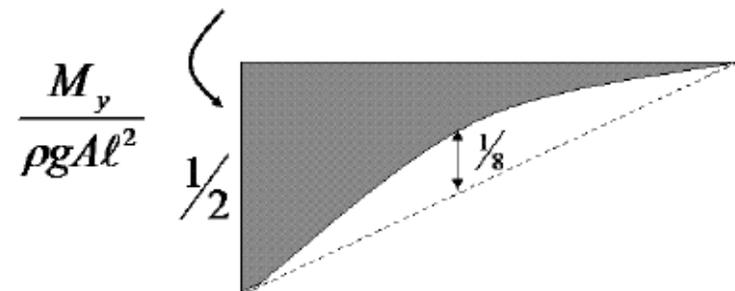
Cantilever beam
Dead weight (gravity)

EQ and solution

$$\forall x \in [0, \ell]; \begin{cases} \frac{dN_x}{dx} = 0 & \Rightarrow N_x = C_1 \\ \frac{dQ_z}{dx} - \rho g A = 0 & \Rightarrow Q_z = \underline{C_3} + \rho g A x \\ \frac{dM_y}{dx} - (C_3 + \rho g S x) = 0 & \Rightarrow M_y = \underline{C_5} + \underline{C_3} x + \frac{1}{2} \rho g A x^2 \end{cases}$$

$$\vec{F}^d(\ell) = 0; \vec{M}^d(\ell) = 0$$

$$Q_z(x) = \rho g A (x - \ell); M_y(x) = \frac{1}{2} \rho g A (\ell - x)^2$$



Example: Coupled M-N strength domain

