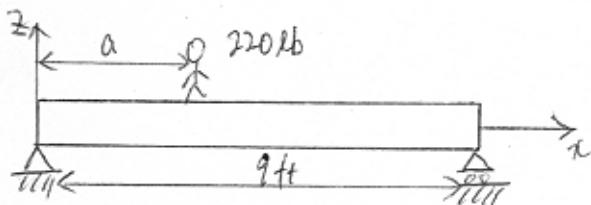
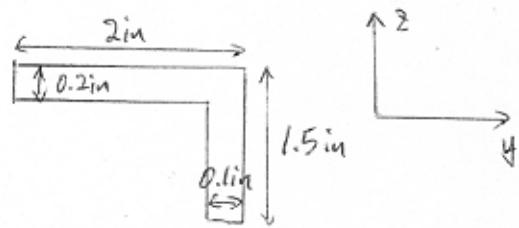


Practice Problems



$$E = 30 \text{ ksi}$$

$$\nu = 0.3$$



The weight of the man on the beam produces a moment about the y-axis only. The moment about the z-axis is zero. Thus, the expression for the deflections in the y and z directions are

$$\frac{d^2w}{dx^2} = \frac{M_y I_z}{E(I_y I_z - I_{yz}^2)} \quad \text{--- (1)}$$

$$\frac{d^2v}{dx^2} = \frac{-M_y I_{yz}}{E(I_y I_z - I_{yz}^2)} \quad \text{--- (2)}$$

(From unit #14, p 20)

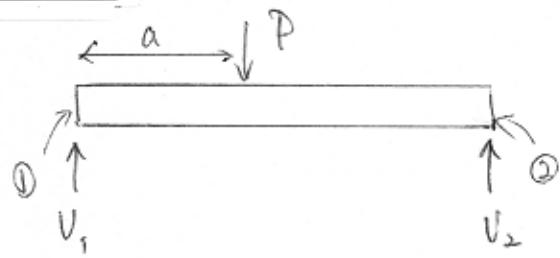
The expression for the in-plane stress, σ_{xx} , is

$$\sigma_{xx} = \frac{M_y (I_{yz} y - I_z z)}{I_y I_z - I_{yz}^2} \quad \text{--- (3)}$$

- We need to determine the maximum total deflection and location of that deflection. In order to calculate these, we first need to obtain the moment, M_y , along the beam, and then the

Second moments of inertia.

Moments



$$\sum F = 0 : V_1 + V_2 = P \quad \left. \right\}$$

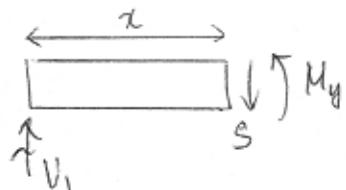
$$\sum M_O = 0 : -Pa + V_2 L = 0 \quad \left. \right\}$$

solving simultaneously, we get

$$V_2 = \frac{Pa}{L}$$

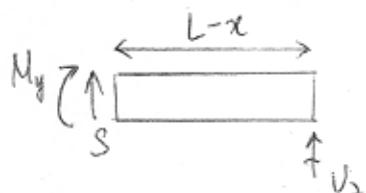
$$V_1 = \frac{P(L-a)}{L}$$

$0 < x < a$:



$$M_y(x) = V_1 x = P \left(1 - \frac{a}{L}\right) x \quad \text{--- (1)}$$

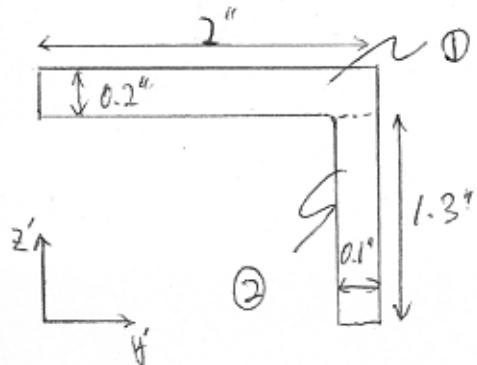
$a < x < L$:



$$M_y(x) = V_2(L-x) = \frac{Pa}{L}(L-x) \quad \text{--- (2)}$$

Moments of Inertia

We need to locate the centroid first.



$$\bar{y} = \frac{\sum A_i \bar{y}_i}{\sum A_i} = \frac{0.65}{0.53} = 1.23$$

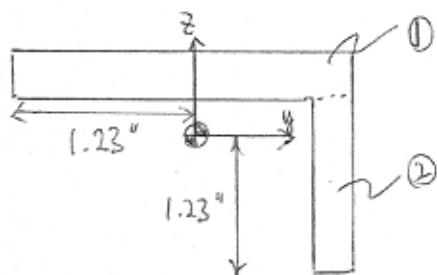
$$\bar{z} = \frac{\sum A_i \bar{z}_i}{\sum A_i} = \frac{0.65}{0.53} = 1.23$$

* values obtained from

Section	$A_i (\text{in}^2)$	$\bar{z}_i (\text{in})$	$A_i \bar{z}_i (\text{in}^3)$	$\bar{y}_i (\text{in})$	$A_i \bar{y}_i (\text{in}^3)$
①	0.4	1.4	0.56	1	0.4
②	0.13	0.65	0.085	1.95	0.25

$$\sum A_i = 0.53 \text{ in}^2 \quad \sum A_i \bar{z}_i = 0.65 \text{ in}^3 \quad \sum A_i \bar{y}_i = 0.65 \text{ in}^3$$

Moving the axes to the centroid, we can now calculate the moments of inertia using



$$I_y = \sum_i (I_{y_i} + A_i z_i^2)$$

$$I_z = \sum_i (I_{z_i} + A_i y_i^2)$$

$$I_{yz} = \sum_i (I_{yz_i} + A_i y_i z_i)$$

I_y :

Section	$b_i (\text{in})$	$h_i (\text{in})$	$\bar{y}_i (\text{in})$	$z_i (\text{in})$	$A_i (\text{in}^2)$	$A_i z_i^2 (\text{in}^4)$
①	2	0.1	1.67 $\times 10^{-4}$	0.17	0.4	0.0116
②	0.1	1.3	0.0183	-0.58	0.13	0.0437
Σ	-	-	0.0185	-	-	0.0553

$$\therefore I_y = 0.0185 + 0.0553$$

$$\Rightarrow I_y = 0.074 \text{ in}^4$$

$I_z:$

Section	$b_i(\text{in})$	$h_i(\text{in})$	$I_{z_i}(\text{in}^4)$	$y_i(\text{in})$	$A_i(\text{in}^2)$	$A_i y_i^2 (\text{in}^4)$
①	0.1	2	0.067	-0.23	0.4	0.0212
②	1.3	0.1	1.08×10^{-4}	0.72	0.13	0.0674
Σ	-	-	0.067	-	-	0.0886

$$\therefore I_z = 0.067 + 0.0886$$

$$\Rightarrow I_z = 0.156 \text{ in}^4$$

$I_{yz}:$

Section	$y_i(\text{in})$	$z_i(\text{in})$	$A_i(\text{in}^2)$	$y_i z_i A_i (\text{in}^4)$
①	-0.23	0.17	0.4	-0.0156
②	0.72	-0.58	0.13	-0.0543
Σ	-	-	-	-0.07

$$\therefore I_{yz} = -0.07 \text{ in}^4$$

Rewriting,

$$I_y = 0.074 \text{ in}^4$$

$$I_z = 0.156 \text{ in}^4$$

$$I_{yz} = -0.07 \text{ in}^4$$

Now, let's find the maximum deflection and location using equations ① and ②. Note that equations ① and ② show that the y and z

derivatives are both proportional to M_y , and they also have the same boundary conditions. Thus, the maximum deflection location for both directions will be identical. Moreover, if we let the coefficients of M_y in equations ① and ② be

$$\frac{1}{I_w} = \frac{I_z}{I_y I_z - I_{y^2}} \quad \text{--- ⑥}$$

$$\frac{1}{I_v} = \frac{I_{y^2}}{I_y I_z - I_{y^2}} \quad \text{--- ⑦}$$

then equations ① and ② become

$$\frac{d^2w}{dx^2} = \frac{M_y}{EI_w} \quad \text{--- ⑧}$$

$$\frac{d^2v}{dx^2} = -\frac{M_y}{EI_v} \quad \text{--- ⑨}$$

Since the form of the differential equations in ⑧ and ⑨ are identical and the boundary conditions are identical, the form of the solution are also identical. Thus, only one of the two equations need to be solved. Let's consider equation ⑧.

OLEICA :

$$\frac{d^2w}{dx^2} = \frac{M_y}{EI_w} = \frac{1}{EI_w} P \left(1 - \frac{a}{L}\right)x \quad (\text{using eq. ④})$$

$$\Rightarrow \frac{dw}{dx} = \frac{P}{EI_w} \left(1 - \frac{a}{L}\right) \frac{x^2}{2} + C_1 \quad \text{--- ⑩}$$

$$\Rightarrow w = \frac{P}{EI_w} \left(1 - \frac{a}{L}\right) \frac{x^3}{6} + C_1 x + C_2 \quad \text{--- (1)}$$

B.C.s ① $x=0 : w=0 \Rightarrow C_2 = 0$

② $x=a :$ $\frac{dw}{dx}$ matches with region $a < x < L$
 w matches with region $a < x < L$ --- (1)

$a < x < L :$

$$\frac{d^2w}{dx^2} = \frac{M}{EI_w} = \frac{Pa}{EI_w L} (L-x) \quad (\text{using eq.(5)})$$

$$\Rightarrow \frac{dw}{dx} = \frac{Pa}{EI_w L} \left(Lx - \frac{x^2}{2}\right) + C_3 \quad \text{--- (14)}$$

$$\Rightarrow w = \frac{Pa}{EI_w L} \left(L\frac{x^2}{2} - \frac{x^3}{6}\right) + C_3 x + C_4 \quad \text{--- (15)}$$

B.C.s ③ $x=L : w=0$

$$\Rightarrow \frac{Pal^2}{2EI_w} - \frac{Pal^2}{6EI_w} + C_3 L + C_4 = 0 \quad \text{--- (16)}$$

④ $x=a :$ same as B.C.'s in ② and ③.

The B.C.s in ② and ③ can be expressed as

$$\begin{aligned} \frac{dw}{dx} \text{ ④ } x=a : \quad & \frac{P}{EI_w} \left(1 - \frac{a}{L}\right) \frac{a^2}{2} + C_1 = \frac{Pa}{EI_w L} \left(La - \frac{a^2}{2}\right) + C_3 \\ \Rightarrow & \frac{a^2}{2} - \frac{a^3}{2L} + C_1 \frac{EI_w}{P} = a^2 - \frac{a^3}{2L} + C_3 \frac{EI_w}{P} \end{aligned} \quad \text{--- (1)}$$

$$\begin{aligned} w \text{ ④ } x=a : \quad & \frac{P}{EI_w} \left(1 - \frac{a}{L}\right) \frac{a^3}{6} + C_1 a = \frac{Pa}{EI_w L} \left(L \frac{a^2}{2} + \frac{a^3}{6}\right) + C_3 a + \\ \Rightarrow & \frac{a^3}{L} - \frac{a^4}{7L} + C_1 a \frac{EI_w}{P} = \frac{a^3}{7} + \frac{a^4}{6L} + C_3 a \frac{EI_w}{P} + C_4 \end{aligned}$$

Our unknowns are C_1 , C_2 and C_4 , and we have three equations ⑩, ⑪ and ⑫. Thus, we can find the three unknowns.

Solving for the unknowns, we get,

$$C_4 = \frac{P}{EI_w} \frac{a^3}{6}$$

$$C_3 = \frac{P}{EI_w} \left(-\frac{a^3}{6L} - \frac{aL}{3} \right)$$

$$C_1 = \frac{P}{EI_w} \left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3} \right)$$

So, the deflections and slopes are

$$0 < x < a : \quad w(x) = \frac{P}{EI_w} \left[\left(1 - \frac{a}{L}\right) \frac{x^3}{6} + \left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3}\right)x \right]$$

$$a < x < L : \quad w(x) = \frac{P}{EI_w} \left[\frac{ax^2}{2} - \frac{ax^3}{6L} + \left(-\frac{a^3}{6L} - \frac{aL}{3}\right)x + \left(\frac{a^3}{6}\right) \right]$$

$$0 < x < a : \quad \frac{dw(x)}{dx} = \frac{P}{EI_w} \left[\left(1 - \frac{a}{L}\right) \frac{x^2}{2} + \left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3}\right) \right]$$

$$\frac{dw(x)}{dx} = \frac{P}{EI_w} \left[ax - \frac{ax^2}{2L} - \frac{a^3}{6L} - \frac{aL}{3} \right]$$

The maximum $w(x)$ occurs when

$$\frac{dw}{dx} = 0 \quad \text{and} \quad \frac{d^2w}{dx^2} = 0 \quad \text{--- ⑬}$$

For $0 < x < a$, equation ⑬ becomes

$$\frac{dw}{dx} = \left(1 - \frac{a}{L}\right) \frac{x^2}{2} + \left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3}\right) = 0 \quad \text{--- ⑭}$$

$$\Rightarrow \frac{x^2}{2} = -\left(\frac{L}{L-a}\right)\left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3}\right) \quad \text{--- (2)}$$

$$\frac{d\omega}{da} = -\frac{x^3}{6L} + \left(a - \frac{a^2}{2L} - \frac{L}{3}\right)x = 0 \quad \text{--- (2)}$$

Substituting equation (2) into equation (2), we get,

$$-\left(\frac{x^2}{2}\right)\left(\frac{x}{3L}\right) + \left(a - \frac{a^2}{2L} - \frac{L}{3}\right)x = 0$$

$$\Rightarrow -\frac{x}{3(L-a)}\left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3}\right) + \left(a - \frac{a^2}{2L} - \frac{L}{3}\right)x = 0$$

$$\Rightarrow x\left[-\frac{1}{3(L-a)}\left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3}\right) + \left(a - \frac{a^2}{2L} - \frac{L}{3}\right)\right] = 0 \quad \text{---}$$

The solution to equation (2) is $x=0$, which is trivial, or

$$-\frac{1}{3(L-a)}\left(\frac{a^2}{2} - \frac{a^3}{6L} - \frac{aL}{3}\right) + \left(a - \frac{a^2}{2L} - \frac{L}{3}\right) = 0$$

$$\Rightarrow \frac{4}{9}\left(\frac{a}{L}\right)^3 - \frac{4}{3}\left(\frac{a}{L}\right)^2 + \frac{11}{9}\left(\frac{a}{L}\right) - \frac{1}{3} = 0 \quad \text{---}$$

The solutions equation (2) are

$$\frac{a}{L} = 1 \text{ or } \frac{1}{2} \text{ or } \frac{3}{2}$$

Since $a=L$ and $a=\frac{3}{2}L$ are not the solution within our bound, a must equal to $\frac{L}{2}$. This result can also be obtained from symmetry arguments. Thus, plugging $a=\frac{L}{2}$ into equation (2), we get

$$\frac{d\omega}{dx} = \frac{x^2}{4} + \left(\frac{\frac{L^2}{8}}{6} - \frac{\frac{L^2}{6}}{8} - \frac{L^2}{6} \right) = 0$$

$$\Rightarrow x^2 = \frac{L^2}{4}$$

$$\therefore x = \pm \frac{L}{2}$$

Since x cannot be $-\frac{L}{2}$, it must be $\frac{L}{2}$. Therefore, the maximum occurs when $a = \frac{L}{2}$ and at $x = \frac{L}{2}$. The deflection at $x = \frac{L}{2}$ can be found by plugging $a = \frac{L}{2}$ into the equation for $0 < x < a$.

$$\omega\left(\frac{L}{2}\right) = \frac{P}{EI_w} \left[\frac{1}{2} \frac{1}{6} \frac{L^3}{8} + \left(-\frac{L^2}{16}\right) \frac{L}{2} \right]$$

$$\Rightarrow \omega\left(\frac{L}{2}\right) = \frac{P}{EI_w} \left(-\frac{L^3}{48}\right) \quad \text{--- (25)}$$

Similarly, the maximum deflection is at $x = \frac{L}{2}$ when $a = \frac{L}{2}$.

$$v\left(\frac{L}{2}\right) = \frac{P}{EI_v} \left(+\frac{L^3}{48}\right) \quad \text{--- (26)}$$

Since

$$I_w = \frac{I_y I_z - I_{yz}^2}{I_2} = \frac{(0.074)(0.156) - (-0.07)^2}{0.156}$$

$$\Rightarrow I_w = 0.0426 \text{ in}^4$$

$$I_v = \frac{I_y I_z - I_{yz}^2}{I_{yz}} = \frac{(0.074)(0.156) - (-0.07)^2}{-0.07}$$

$$\Rightarrow I_v = -0.0949 \text{ in}^4$$

Therefore, the values for $w(\frac{L}{2})$ and $v(\frac{L}{2})$ are:

$$w\left(\frac{L}{2}\right) = \frac{220 \text{ lb}}{(30 \text{ ksi})(0.0426 \text{ in}^4)} \left(-\frac{(9x12)^3}{48} \right)$$

$$\therefore w\left(\frac{L}{2}\right) = -4.52 \text{ in}$$

$$v\left(\frac{L}{2}\right) = \frac{220 \text{ lb}}{(30 \text{ ksi})(0.0949 \text{ in}^4)} \left(+\frac{(9x12)^3}{48} \right)$$

$$\therefore v\left(\frac{L}{2}\right) = -2.03 \text{ in}$$

b) From equation ③, the stress, σ_{xx} is

$$\sigma_{xx} = \frac{M_y(T_{yz}y - T_{zx}z)}{I_y I_z - I_{yz}^2} = \frac{(-0.07y - 0.156z)}{(0.074)(0.156) - (-0.07)^2} M_y.$$

For $0 < x < a$, M_y is maximum at $x = \frac{L}{2}$ when $a = \frac{L}{2}$ (see equation)

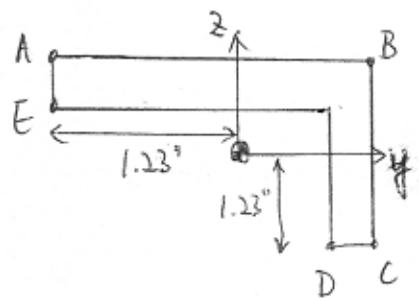
Thus, σ_{xx} is maximum at

$$\sigma_{xx} = \frac{(-0.07y - 0.156z)}{0.0066} \frac{PL}{4} = \frac{(220 \text{ lb})(4x12)}{4}$$

$$\Rightarrow \sigma_{xx} = -6.3 \times 10^4 y + 14 \times 10^4 z \text{ (psi)}$$

σ_{xx} is larger at points that are further away from the ne

axis.



Point	$f_i(\text{in})$	$z_i(\text{in})$	$\sigma_{zx}(\text{psi})$
A	-1.23	0.27	11.5×10^4
B	0.77	0.27	-1.1×10^4
C	0.77	-1.23	-22×10^4
D	0.67	-1.23	-21×10^4
E	-1.23	0.07	8.7×10^4

Thus, from the table, the maximum value is

$$\sigma_{zx\max} = -22 \times 10^4 \text{ psi}$$

④ point C.