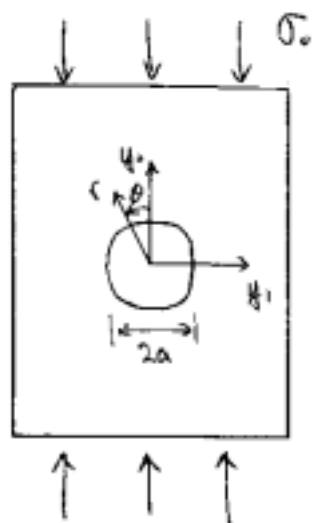


Solutions to Home Assignment #4

Warm-Up Exercises

1. To find the expressions for the stress components at the hole boundary as a function of θ , we will use the Airy stress functions. The problem we need to consider is an isotropic plate with a hole under compressive load.



From our notes (unit 8, p.20), our assumed stress function, ϕ , for an isotropic plate with a hole in polar coordinates is

$$\begin{aligned}\phi(r, \theta) = & [A_0 + B_0 \ln r + C_0 r^2 + D_0 r^3 \ln r] \\ & + [A_2 r^2 + \frac{B_2}{r} + C_2 r^4 + D_2] \cos 2\theta \quad - ①\end{aligned}$$

In order for the displacements to be single-valued, D_0 is set to equal to zero. For the stresses to be bounded as $r \rightarrow \infty$, C_2 also needs to be zero. The stress in polar coordinates can now be expressed as:

$$\begin{aligned}\sigma_{rr} &= \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} \\ &= \frac{B_0}{r^2} + 2C_0 - [2A_2 + \frac{6B_2}{r^4}] \cos 2\theta \quad \text{--- (2)}\end{aligned}$$

$$\begin{aligned}\sigma_{\theta\theta} &= \frac{\partial^2 \phi}{\partial r^2} \\ &= -\frac{B_0}{r^2} + 2C_0 + [2A_2 + \frac{6B_2}{r^4}] \cos 2\theta \quad \text{--- (3)}\end{aligned}$$

$$\begin{aligned}\sigma_{r\theta} &= -\frac{1}{r} \frac{\partial^2 \phi}{\partial r \partial \theta} + \frac{1}{r^2} \frac{\partial \phi}{\partial \theta} \\ &= [2A_2 - \frac{6B_2}{r^4} - \frac{2D_2}{r^2}] \sin 2\theta \quad \text{--- (4)}\end{aligned}$$

The boundary conditions are:

$$\sigma_{rr} = \sigma_{r\theta} = 0 \quad \text{--- (5)} \quad r = a \quad \leftarrow \text{stress-free at hole edge.}$$

$$\sigma_{\theta\theta} = \sigma_0, \quad \sigma_{r\theta} = 0 \quad \text{--- (6)} \quad \theta_1 \rightarrow \infty$$

$$\sigma_{xx} = 0, \quad \sigma_{xy} = 0 \quad \text{--- (7)} \quad \theta_1 \rightarrow \infty$$

Using these boundary conditions, we can find the unknown constants in equations ② through ④. Skipping the math

(as described in notes)^{unit 8, p. 26}, we can get

$$\sigma_{rr} = \frac{\sigma_0}{2} \left(1 - \frac{a^2}{r^2} \right) + \frac{\sigma_0}{2} \left(1 - 4 \frac{a^2}{r^2} + 3 \frac{a^4}{r^4} \right) \cos 2\theta \quad \text{--- ⑤}$$

$$\sigma_{\theta\theta} = \frac{\sigma_0}{2} \left(1 + \frac{a^2}{r^2} \right) - \frac{\sigma_0}{2} \left(1 + 3 \frac{a^2}{r^2} \right) \cos 2\theta \quad \text{--- ⑥}$$

$$\sigma_{r\theta} = - \frac{\sigma_0}{2} \left(1 + 2 \frac{a^2}{r^2} - 3 \frac{a^4}{r^4} \right) \sin 2\theta \quad \text{--- ⑦}$$

At the hole boundary, i.e., $r=a$, the stresses are reduced to

$$\sigma_{rr} = 0 \quad \text{--- ⑧}$$

$$\sigma_{\theta\theta} = - \sigma_0 (1 - 2 \cos 2\theta) \quad \text{--- ⑨}$$

$$\sigma_{r\theta} = 0 \quad \text{--- ⑩}$$

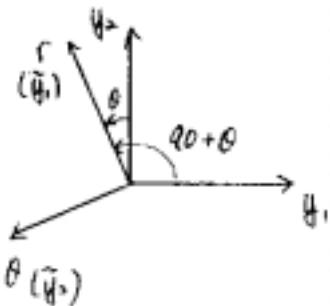
Now, we need to transform our stresses from the polar to Cartesian system.

Set

$$\tilde{\sigma}_{ii} = \tilde{\sigma}_{rr} = 0$$

$$\tilde{\sigma}_{zz} = \tilde{\sigma}_{\theta\theta}$$

$$\tilde{\sigma}_{12} = \tilde{\sigma}_{r\theta} = 0$$



To transform our r, θ (i.e. $\hat{y}_1, -\hat{y}_2$) system to our y_1, y_2 system, we must rotate through an angle $-(90 + \theta)$.

The transformation rule is

$$\sigma_{\alpha\beta} = l_{\alpha i} l_{\beta j} \tilde{\sigma}_{ij} \quad \text{--- --- ⑪}$$

This equation reduces to

$$\sigma_{ii} = l_{1i} l_{1i} \tilde{\sigma}_{ii} = \cos^2 \theta \tilde{\sigma}_{ii} \quad \text{--- --- ⑫}$$

$$\sigma_{zz} = l_{22} l_{22} \tilde{\sigma}_{zz} = \cos^2(90 + \theta) \tilde{\sigma}_{zz} = \sin^2 \theta \tilde{\sigma}_{zz} \quad \text{--- ⑬}$$

$$\sigma_{ii} = l_{1i} l_{2i} \tilde{\sigma}_{zz} = (-\cos \theta)(-\sin \theta) \tilde{\sigma}_{zz} = \sin \theta \cos \theta \tilde{\sigma}_{zz} \quad \text{--- ⑭}$$

$$\star \quad l_{1i} = \cos(-180 - \theta) = -\cos \theta$$

$$l_{2i} = \cos(-90 - \theta) = -\sin \theta$$

Therefore, plugging the expression for $\sigma_{\theta\theta}$ in equations ⑫ through ⑯, we get :

$$\sigma_{rr} = -\sigma_0 (1 - 2\cos 2\theta) \cos^2 \theta$$

$$\sigma_{\theta\theta} = -\sigma_0 (1 - 2\cos 2\theta) \sin^2 \theta$$

$$\sigma_{rz} = \sigma_0 (1 - 2\cos 2\theta) \sin \theta \cos \theta$$

For use in problems 2 and 3, the stress can be expressed in normalized form as :

y_1, y_2 coordinate : $\sigma_{rr}/\sigma_0 = -(1 - 2\cos 2\theta) \cos^2 \theta$

$$\sigma_{\theta\theta}/\sigma_0 = -(1 - 2\cos 2\theta) \sin^2 \theta$$

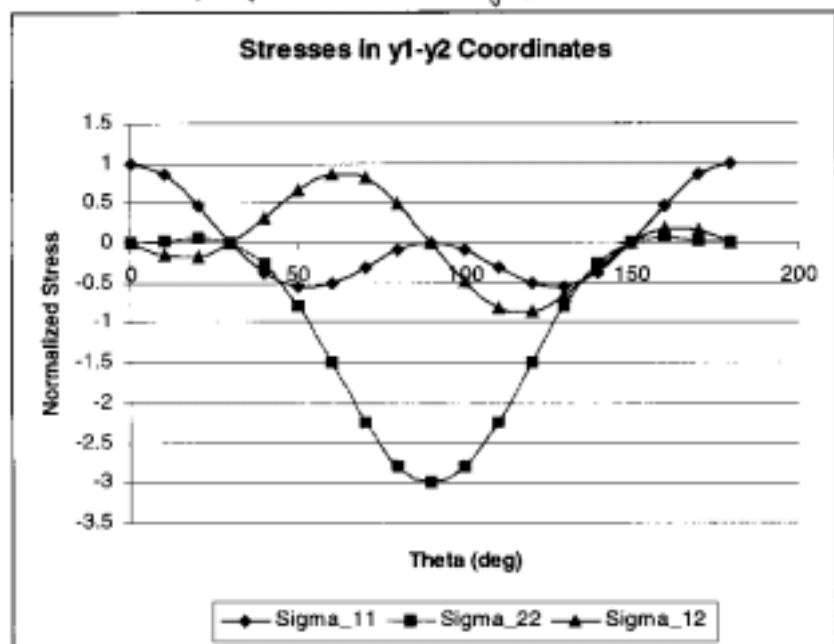
$$\sigma_{rz}/\sigma_0 = (1 - 2\cos 2\theta) \sin \theta \cos \theta$$

$r-\theta$ coordinate : $\sigma_{rr}/\sigma_0 = 0$

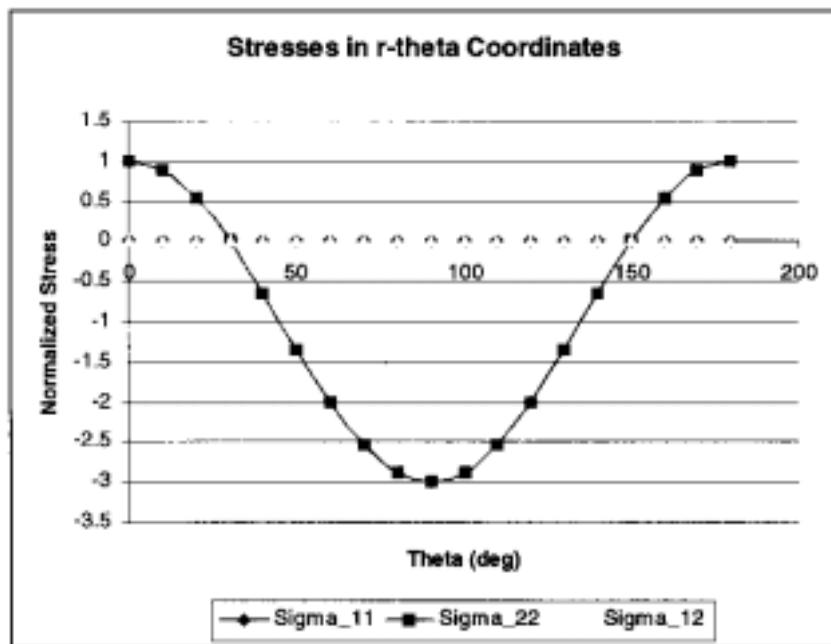
$$\sigma_{\theta\theta}/\sigma_0 = -(1 - 2\cos 2\theta)$$

$$\sigma_{rz}/\sigma_0 = 0$$

2. Stresses in y_1 - y_2 coordinate system



3. Stresses in r - θ coordinate system



4. We can derive a number of interesting results from these plots.

- ① Extensional stresses (σ_{rr} , $\sigma_{\theta\theta}$, σ_{zz}) are symmetric about each 90° rotation.
- ② Shear stresses (σ_{rz}) are anti-symmetric about each 90° rotation.
- ③ All stresses go to zero at $\theta = 30^\circ$, 150° , 210° and 330° .
- ④ σ_{rr} and $\sigma_{\theta\theta}$ are zero at hole edge. Only σ_{zz} is non-zero.
- ⑤ In the plots, all stresses are normalized by σ_v , which is negative in the present case. So, the actual stress state has the same shape but opposite sign.