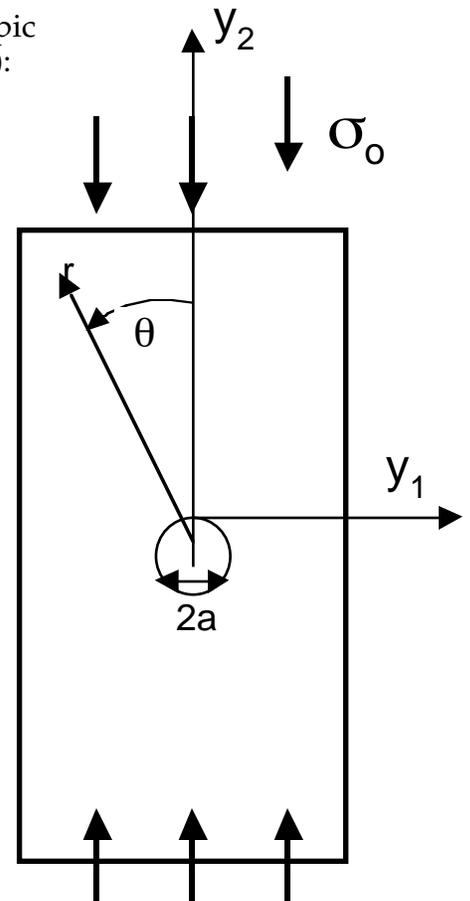


## HOME ASSIGNMENT #4

### Warm-Up Exercises

Let's consider the stress field around the hole in an isotropic plate loaded in uniaxial compression (no buckling occurs):

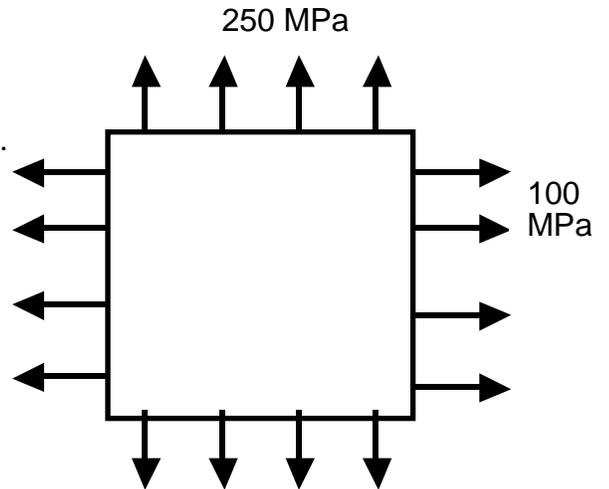
1. Derive expressions for the stress components at the hole boundary ( $r = a$ ) as a function of  $\theta$  in the  $y_1 - y_2 - y_3$  coordinate system.
2. Plot these values, normalized by the applied far-field stress,  $\sigma_0$ , as a function of  $\theta$ .
3. Plot the "polar coordinate" stresses at the hole boundary as a function of  $\theta$ .
4. Compare the plots of #2 and #3. Discuss any similarities/differences/noteworthy points.



### Practice Problems

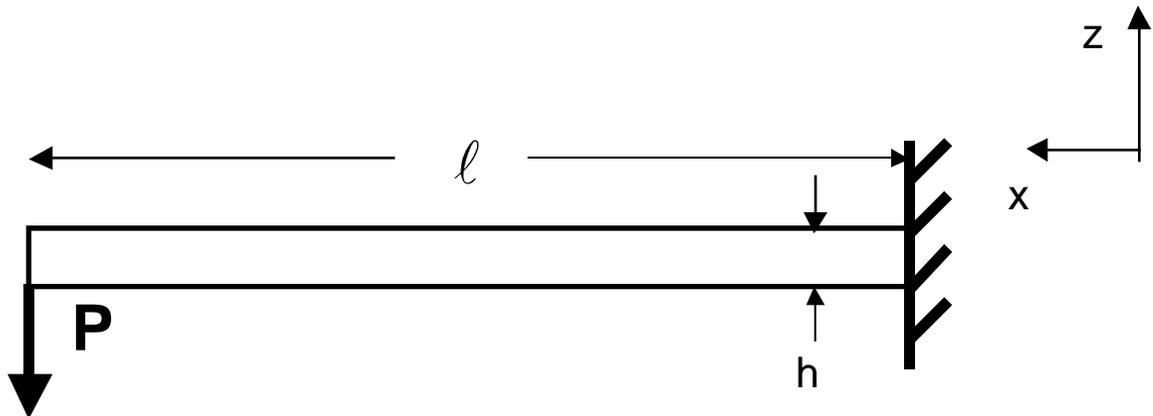
5. A pressurized aluminum vessel results in a state of plane stress in the skin of 100 MPa in one direction and 250 MPa perpendicular to this direction. Aluminum has a modulus of approximately 70 GPa and a Poisson's ratio of 0.30. Using the stress function approach:

- (a) Construct a solution for this configuration.
- (b) Determine the full strain state.
- (c) Determine the entire deformation field.



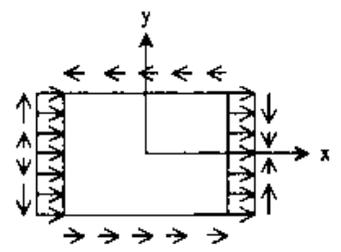
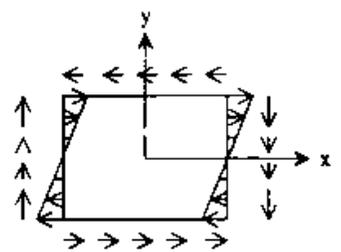
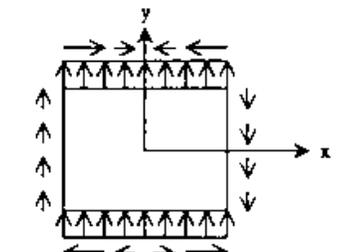
### Application Tasks

6. Use appropriate combinations of the Airy stress functions provided to:
- (a) Construct a solution for the case of an isotropic cantilevered beam loaded by an end load as pictured.
  - (b) Compare these results (for  $\sigma_{xx}$  and  $\sigma_{xz}$ ) with those from simple beam theory.
  - (c) Where in this problem is it necessary to apply St. Venant's principle?



$$\phi = C_{ij} x^i y^j$$

Stress function $\phi$	$\sigma_{xx}$	$\sigma_{yy}$	$\sigma_{xy}$	Surface Forces on Rectangle
$C_{20} x^2$	0	$2 C_{20}$	0	
$C_{30} x^3$	0	$6 C_{30} x$	0	
$C_{02} y^2$	$2 C_{02}$	0	0	
$C_{03} y^3$	$6 C_{03} y$	0	0	
$C_{11} x y$	0	0	$-C_{11}$	

$C_{12} x y^2$	$2 C_{12} x$	$0$	$-2 C_{12} y$	
$C_{13} x y^3$	$6 C_{13} x y$	$0$	$-3 C_{13} y^2$	
$C_{21} x^2 y$	$0$	$2 C_{21} y$	$-2 C_{21} x$	
$C_{31} x^3 y$	$0$	$6 C_{31} x y$	$-3 C_{31} x^2$	