

**Block 2 Stress and Strain**

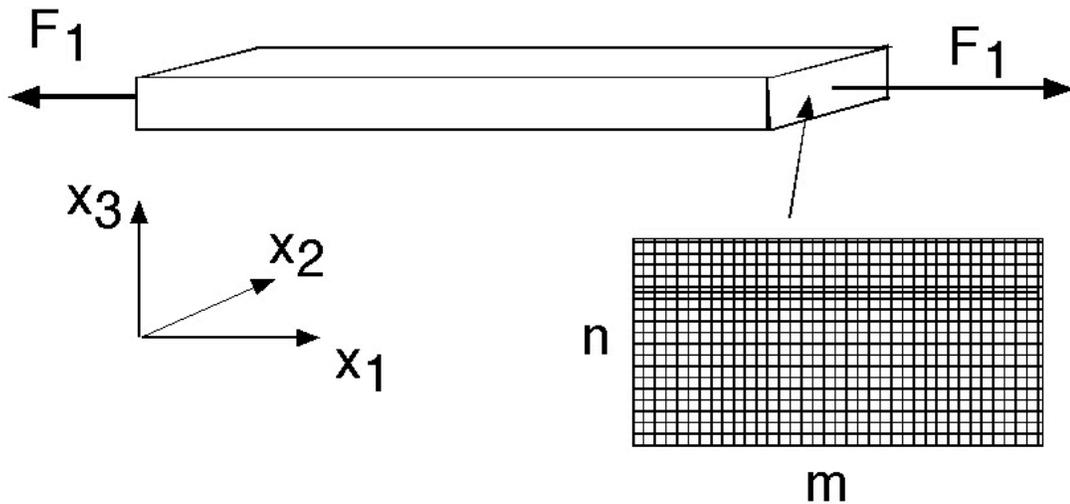
**M10 Stress**

We want to look more closely at how structures transmit loads. In general, structures are not made up of discrete, 1-D bars, but continuous, complex, 3-D shapes. If we want to understand how they transmit load we need more tools. To this end we will introduce the concept of stress (and later strain). We will also apply three great principles in terms of stress and strain.

Definition Stress is the measure of the intensity (per area) of force acting at a point.

Use new coordinate system  $x, y, z \square x_1, x_2, x_3$

Example:



Divide  $x_1$  face into  $m \times n$  squares

On a 2<sup>nd</sup> square

$$\square F_1 = F_1 / nm$$

$$\square A = A / nm$$

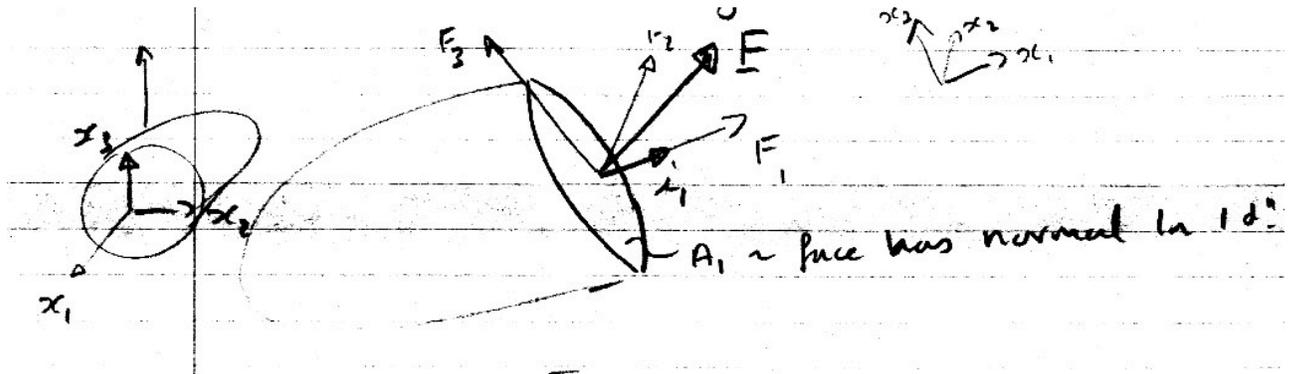
The stress is the intensity, force per unit area, so let  $m$  and  $n$  go to infinity.

$$\sigma_1 = \lim_{\substack{m \rightarrow \infty \\ n \rightarrow \infty}} \frac{\Delta F_1}{\Delta A} \quad \text{units } \frac{\text{force}}{\text{length}^2}$$

where  $\sigma_1$  is the stress at a point on the  $x_1$  - face

- Magnitude  $\sigma_1$
- Direction  $i_1$

Consider a more general case:



Force  $\underline{F}$  acting on  $x_1$  face ( $i_1$  is normal to plane of face)

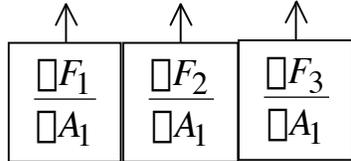
Resolve force into 3 components

$$\underline{F} = F_1 i_1 + F_2 i_2 + F_3 i_3$$

Then take the limit as the force on the face is carried by a smaller set of areas (bigger grid).

$$\lim_{\Delta A \rightarrow 0} \frac{\Delta \underline{F}}{\Delta A} = 0 \quad (\text{but stress vector has other components})$$

$$\underline{\sigma} = \sigma_{11} \underline{i}_1 + \sigma_{12} \underline{i}_2 + \sigma_{13} \underline{i}_3$$



Where  $\underline{\sigma}_1$  is the stress vector and  $\sigma_{1n}$  are the components acting on the  $x_1$  normal face, in the  $x_1, x_2$  and  $x_3$  directions.

Get similar results if we looked at the  $x_2$  and  $x_3$  faces.

$$\underline{\sigma}_2 = \sigma_{21} \underline{i}_1 + \sigma_{22} \underline{i}_2 + \sigma_{23} \underline{i}_3 \quad \text{on } i_2 \text{ face}$$

$$\underline{\sigma}_3 = \sigma_{31} \underline{i}_1 + \sigma_{32} \underline{i}_2 + \sigma_{33} \underline{i}_3 \quad \text{on } i_3 \text{ face}$$

can write more succinctly as

$$\underline{\sigma}_m = \sigma_{mn} \underline{i}_n$$

$\sigma_{mn}$  is the stress tensor,  $\underline{\sigma}_m$  is the stress vector.

This method of representing a set of vector equations is an example of indicial, or tensor, notation:

### Tensor (Indicial) Notation

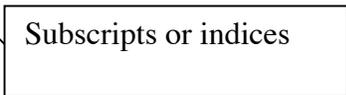
"easy" to write complicated formulae

"easy" to mathematically manipulate

"elegant" rigorous

Example:

$$x_i = F_{ij} Y_j$$



### By Convention

Latin subscripts  $m, n, i, j, p, q, \text{etc.}$

Take values 1, 2, 3, (3-D)

Greek Subscripts  $\square, \square, \square$

Take values 1 or 2 (2-D)

**3 Rules for Tensor Manipulation**

1. A subscript occurring twice is a repeated (or dummy) index and is summed over 1, 2, (and) 3.

- Implicit summation

$$\text{For } x_i = F_{ij}Y_j = \sum_{j=1}^3 F_{ij}Y_j$$

i.e:  $x_i = F_{i1}Y_1 + F_{i2}Y_2 + F_{i3}Y_3$

2. A subscript occurring once in a term is called a free index, can take on the range 1, 2, (3) but is not summed. It represents separate equations.

$$x_1 = F_{11}Y_1 + F_{12}Y_2 + F_{13}Y_3 \quad (\text{cf matrix multiplication})$$

$$x_2 = F_{21}Y_1 + F_{22}Y_2 + F_{23}Y_3$$

$$x_3 = F_{31}Y_1 + F_{32}Y_2 + F_{33}Y_3$$

3. No index can appear in a term more than twice.

**Two Useful Tensor Parameters**

**1. Kronecker delta**

$$\square_{mn} = \begin{cases} 1 & \text{when } m=n \\ 0 & \text{when } m \neq n \end{cases}$$

Why is this useful?

Consider scalar product of unit vectors

$$i_m \cdot i_m = 1 \text{ parallel}$$

$$i_m \cdot i_n = 0 \text{ perpendicular}$$

$$\square_{mn} = i_m \cdot i_n$$

So Scalar product of two vectors becomes:

$$\begin{aligned} \underline{F} \cdot \underline{G} &= (F_m \underline{i}_m) \cdot G_n \underline{i}_n \\ &= F_m G_n (\underline{i}_m \cdot \underline{i}_n) \\ &= F_m G_n \delta_{mn} \end{aligned}$$

2. **Permutation tensor**  $e_{mnp}$ . ("permute" - to change the order of)

$$e_{mnp} = \begin{cases} 0 & \text{When any two indices are equal} \\ 1 & \text{When } mnp \text{ is an \underline{even} permutation of } 1, 2, 3 \\ -1 & \text{When } mnp \text{ is an \underline{odd} permutation of } 1, 2, 3 \end{cases}$$

Swapped 2 pairs  
 1, 2, 3  
 3, 1, 2  
 2, 3, 1

Swapped 1 pair  
 1, 3, 2  
 2, 1, 3  
 3, 2, 1

(So where does this come from?)

Consider cross product of unit vectors.

$$\begin{aligned} \underline{i}_1 \times \underline{i}_1 &= 0 & \underline{i}_1 \times \underline{i}_2 &= \underline{i}_3 & \underline{i}_2 \times \underline{i}_1 &= -\underline{i}_3 \\ \underline{i}_2 \times \underline{i}_2 &= 0 & \underline{i}_2 \times \underline{i}_3 &= \underline{i}_1 & \underline{i}_3 \times \underline{i}_2 &= -\underline{i}_1 \\ \underline{i}_3 \times \underline{i}_3 &= 0 & \underline{i}_3 \times \underline{i}_1 &= \underline{i}_2 & \underline{i}_1 \times \underline{i}_3 &= -\underline{i}_2 \end{aligned}$$

So  $\underline{i}_1 \times \underline{i}_2 = e_{121}\underline{i}_1 + e_{122}\underline{i}_2 + e_{123}\underline{i}_3 = \underline{i}_3$

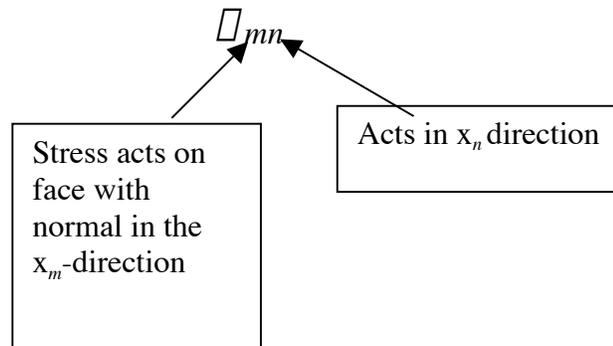
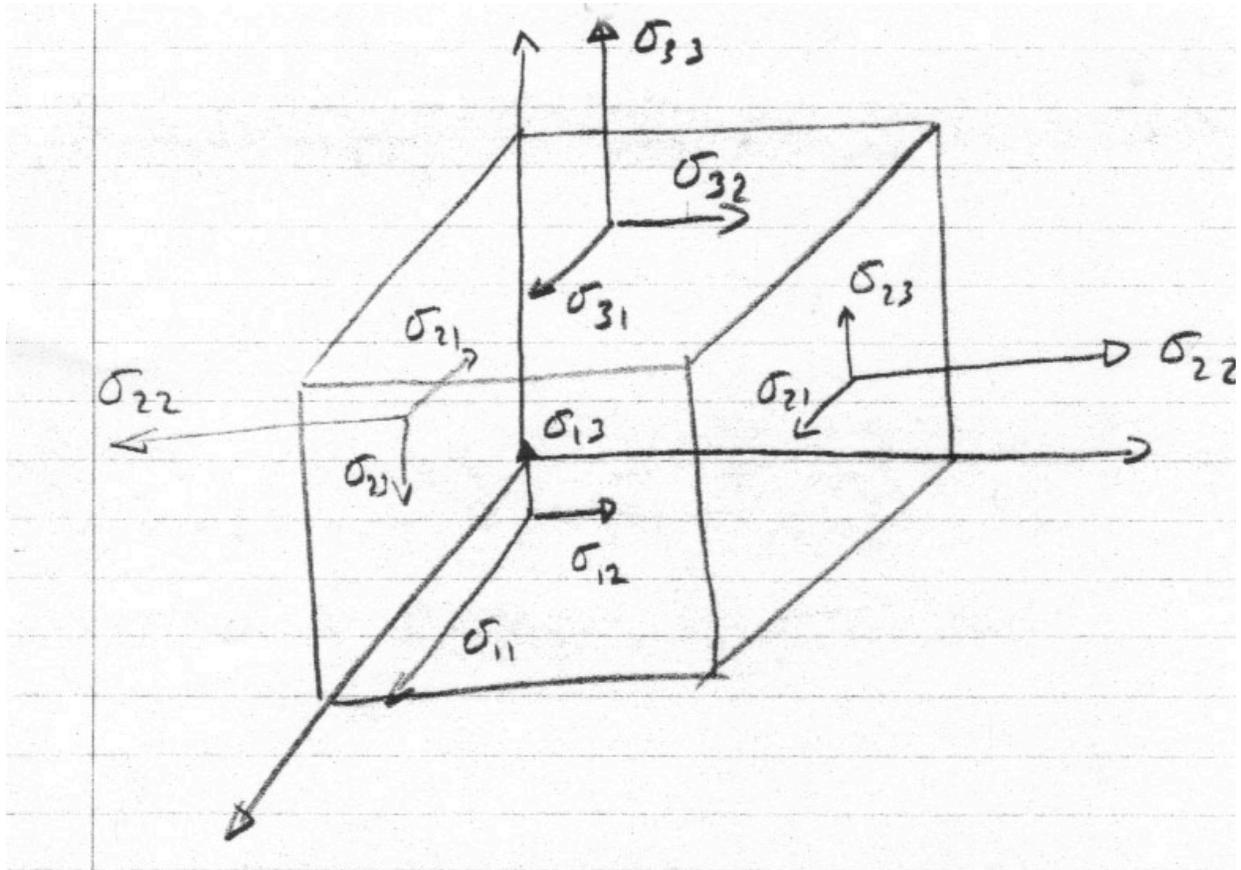
In general,

Now back to stress. We have learnt that stress is a tensor quantity, i.e. it can be represented by tensor (indicial) notation

**Stress Tensor and Stress Types**

$\sigma_{mn}$  is the stress tensor

Consider a differential element:



Types of Stress

Can identify two distinct types of stress:

Normal (Extensional) - acts normal to face

$\sigma_{11}, \sigma_{22}, \sigma_{33}$  - acts to extend element

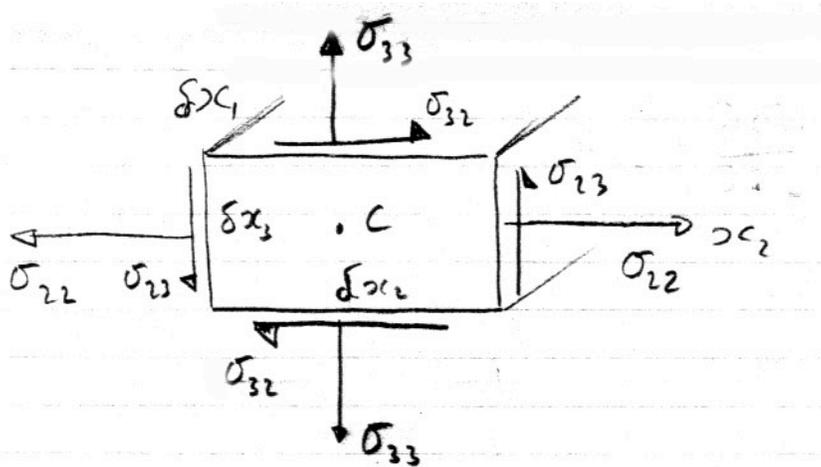
Shear Stress - acts parallel to face plane

$\sigma_{12}, \sigma_{13}, \sigma_{21}, \sigma_{23}, \sigma_{31}, \sigma_{32}$  acts to shear element

i.e. 9 components of stress, but it turns out that not all of these are independent:

Symmetry of Stress Tensor

Consider moment equilibrium of differential element:



Take moments about C  $\curvearrowright +$

$$2\sigma_{23} \delta x_3 \delta x_1 \left(\frac{\delta x_2}{2}\right) - 2\sigma_{32} \delta x_2 \delta x_1 \left(\frac{\delta x_3}{2}\right) = 0$$

simplifies to:  $\sigma_{23} = \sigma_{32}$

Therefore only 6 independent components of the stress tensor: three extensional stresses:

$\sigma_{11}, \sigma_{22}, \sigma_{33}$  and three shear stresses:  $\sigma_{12}, \sigma_{23}, \sigma_{31}$